

Supplementary Information

# **Thermo-Adaptive Interfacial Solar Evaporation Enhanced by Dynamic Water Gating**

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## **Outline**

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### **Supplementary Method 1. DSC Measurement of Vaporization Enthalpy**

We measured the evaporation enthalpy of water in evaporators using differential scanning calorimetry. The vaporization enthalpy measurements were carried out by placing the saturated foam in an open aluminum crucible and subjecting it to a linear heating rate of  $5 \text{ K min}^{-1}$  under a nitrogen flow rate of  $20 \text{ mL min}^{-1}$ , within a temperature range of 20 to  $180 \text{ }^\circ\text{C}$ .

## **Supplementary Method 2. Lithium-ion Solvation Experiment**

We prepared a LiCl water solution varying the concentration in  $50 \text{ g L}^{-1}$  and  $100 \text{ g L}^{-1}$ , and the evaporated vapor was condensed and collected. The  $\text{Li}^+$  concentration in the collected water was measured using Inductively Coupled Plasma Mass Spectrometry (ICP-MS). To verify the formation of water clusters during evaporation, we utilized the solvation properties of Lithium-ion ( $\text{Li}^+$ ) in a water collection experiment, demonstrating enhanced evaporation through water cluster formation.

### Supplementary Note 1. Surface Free Energy Calculation

Surface free energy ( $\gamma$ ) was composed of various inter-molecular attractive forces, including polar components ( $\gamma_s^p$ ) and dispersive components ( $\gamma_s^d$ ) at the interface. Based on the contact angle data and fitted according to the Owens, Wendt, Rabel and Kaelble (OWRK) method<sup>1</sup>, the  $\gamma$  was calculated using the following equation (1):

$$1 + \cos \theta = 2 \sqrt{\gamma_s^d} \left( \frac{\sqrt{\gamma_l^d}}{\gamma_{lv}} \right) + 2 \sqrt{\gamma_s^p} \left( \frac{\sqrt{\gamma_l^p}}{\gamma_{lv}} \right) \quad (1)$$

where  $\theta$  was the contact angle of liquid, the superscripts p and d corresponding to the polar and dispersion force components, respectively.  $\gamma_{lv}$  was the free energy between liquid and solid against their saturated vapor, while  $\gamma_l^d$  and  $\gamma_l^p$  referred to the free energies of liquids as reported in many references<sup>2,3</sup>.

## Supplementary Note 2. Energy conversion efficiency

The solar energy conversion efficiency ( $\eta$ ) is calculated by the equation:

$$\eta = mh_v/C_{opt}P_0 \quad (2)$$

where  $m$  represents the net evaporation rate calculated by subtracting evaporation rate under dark condition from evaporation rate under one sun irradiation,  $h_v$  refers to the equivalent evaporation enthalpy of water in SDWEs.  $P_0$  is the solar irradiation power of one sun ( $1 \text{ kW m}^{-2}$ ).  $C_{opt}$  refers to the optical concentration on evaporator surface.

For p-SDWE,  $m = 3.58 - 0.45 = 3.13 \text{ kg m}^{-2} \text{ h}^{-1}$ ,  $E_{eq} = 1080 \text{ J g}^{-1}$ . Therefore, the solar energy conversion efficiency is 93.9% under one sun illumination.

### Supplementary Note 3. Energy loss analysis

The energy loss during the entire process can be attributed to four main factors: conduction, radiation, convection, and reflection. Each heat loss parameters were calculated as follows:

#### (1) Conduction

$$Q_{\text{conduction}} = CmD_T$$

where  $Q_{\text{conduction}}$  referred to the heat energy transfer from the evaporator to bulk water,  $C$  was the specific heat capacity of pure water ( $4.2 \text{ kJ kg}^{-1} \text{ K}^{-1}$ ),  $m$  was the weight of water and  $D_T$  is the temperature change of bulk water under one sun irradiation within 3600 s. Therefore, the conduction heat loss of the evaporator was  $\sim 1.86\%$  under 1 sun irradiation.

#### (2) Radiation

$$Q_{\text{radiation}} = \varepsilon A \sigma (T_s^4 - T_0^4)$$

where  $\varepsilon$  corresponding to the emissivity (assumed to be 1),  $A$  was the evaporator area ( $5.0 \times 5.0 \text{ cm}^2$ ), and  $\sigma$  was the Stefan-Boltzmann constant ( $5.669 \times 10^{-8} \text{ W m}^{-2} \text{ K}^{-4}$ ),  $T_s$  was associated to the temperature of evaporator surface,  $T_0$  was the ambient water temperature. The surface radiation heat loss accounts for  $\sim 2.84\%$  of the total received energy.

#### (3) Convection

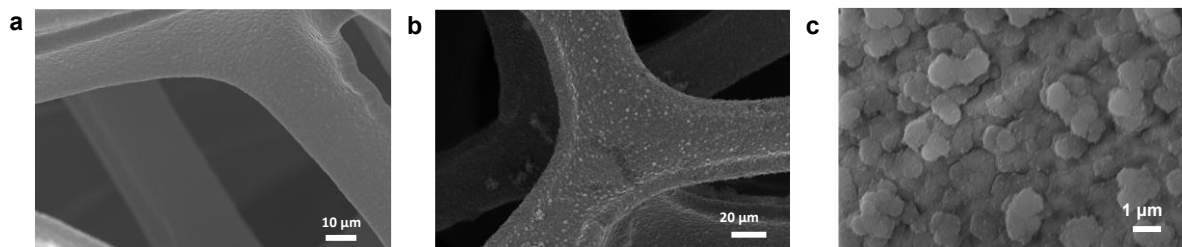
$$Q_{\text{convection}} = hA (T_s - T_0)$$

where  $h$  was the convection heat transfer coefficient ( $5 \text{ W m}^{-2} \text{ K}^{-1}$ ). The convection heat loss made up for  $1.9\%$  of the total received energy.

#### (4) Reflection

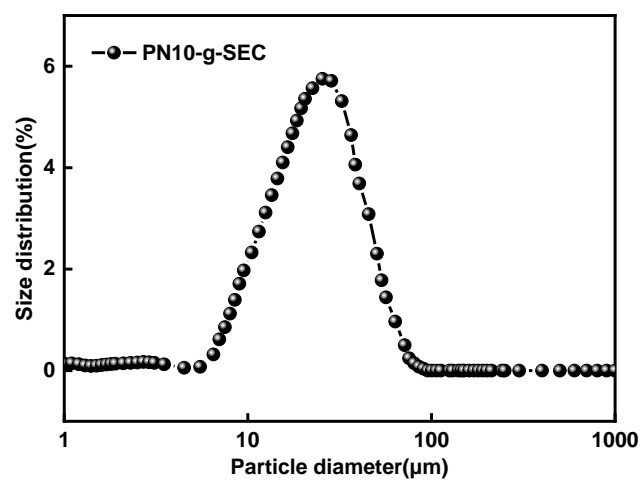
The measured average reflection loss of p-DSF over the broad solar spectrum (200–2500 nm) was  $\sim 5.6\%$ .

Therefore, the sum of calculated energy losses is  $12.2\%$  under 1 sun irradiation.

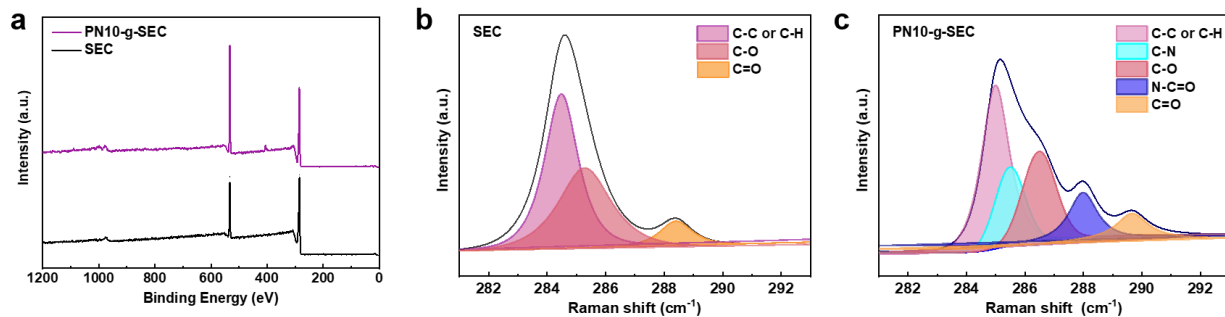


**Supplementary Figure 1.** The morphology of PDA layer: **a.** s-PDA, **b.** r-PDA, **c.** The PDA cluster on r-PDA layer.

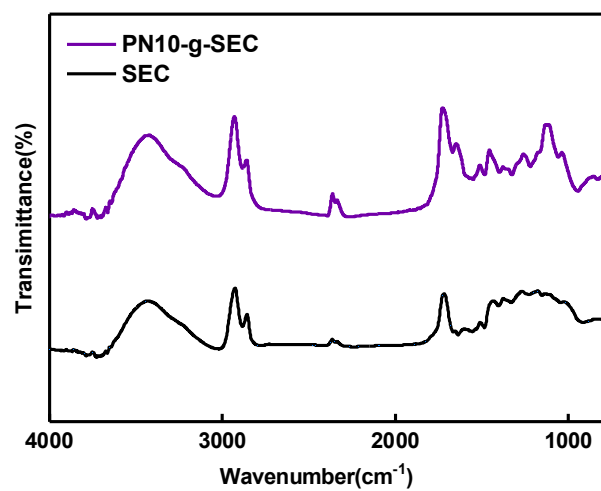




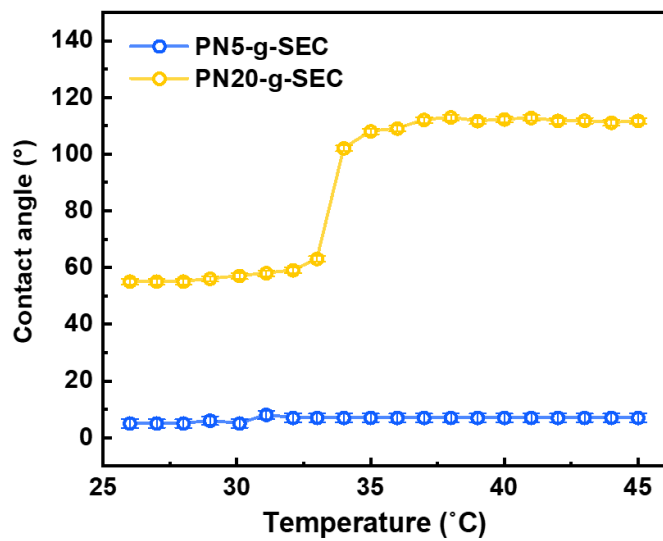
**Supplementary Figure 2.** Particle Size distribution of PN10-g-SEC.



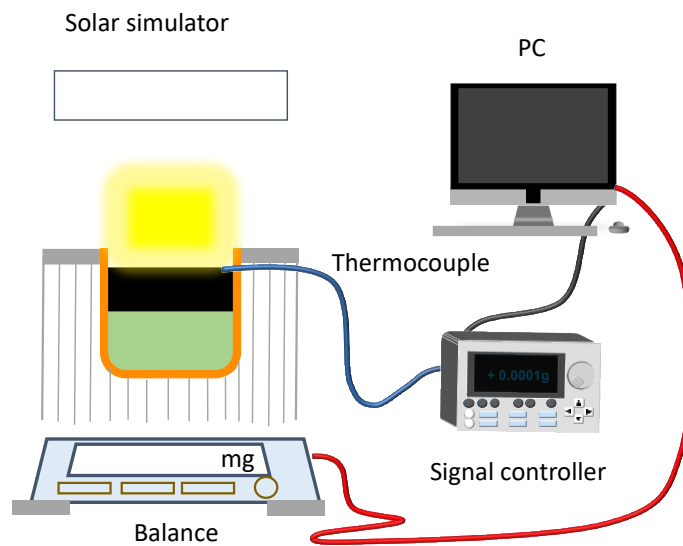
**Supplementary Figure 3.** XPS spectrum of PN10-g-SEC: **a.** XPS characterization of PN10-g-SEC and SEC, **b.** C1s fitting spectra of SEC, **c.** C1s fitting spectra of PN10-g-SEC



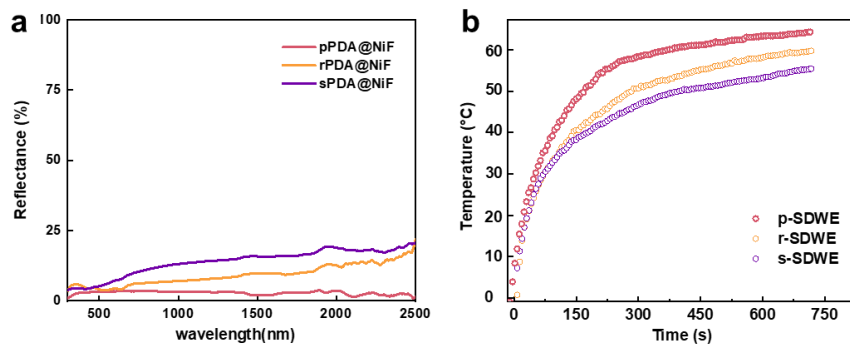
**Supplementary Figure 4.** FT-IR spectrum of SEC and PN10-g-SEC.



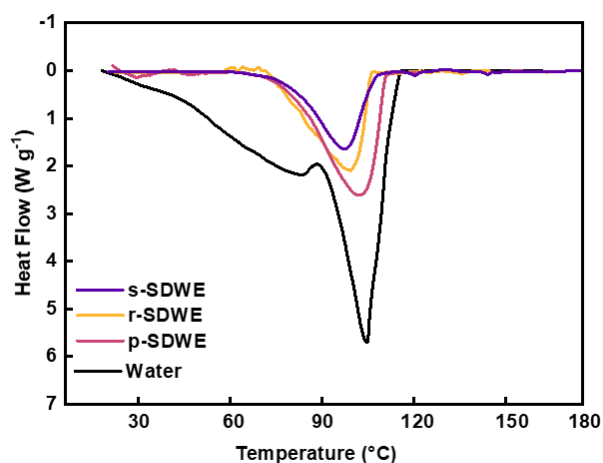
**Supplementary Figure 5.** Contact angle variation of PN5-g-SEC and PN20-g-SEC under different temperature



**Supplementary Figure 6.** The set-up for measuring the solar evaporation performance

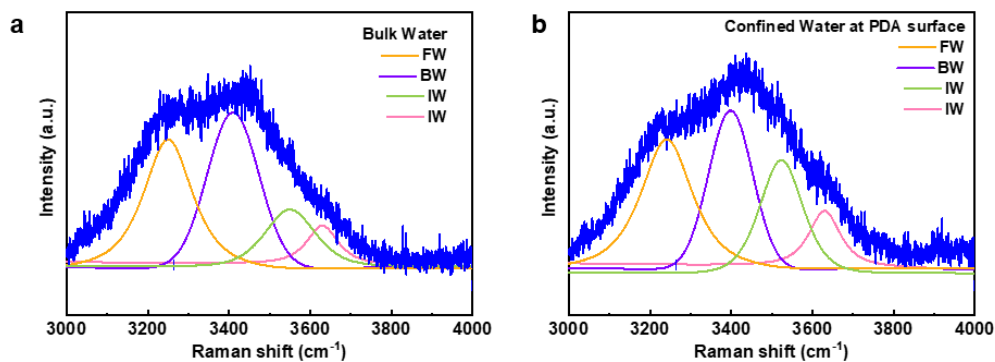


**Supplementary Figure 7. a.** Reflectance of PDA layer, comprising of pPDA@NiF, rPDA@NiF, sPDA@NiF, **b.** The surface temperature of dried SDWEs under solar illumination of  $1\text{KW m}^{-2}$



**Supplementary Figure 8.** Thermograms of pure water and SDWEs.

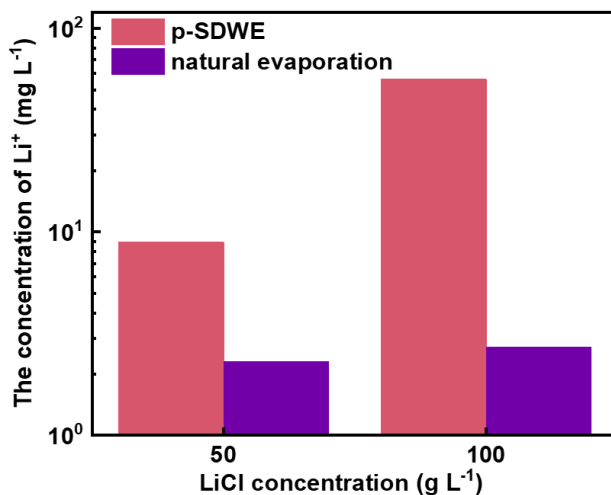
The effective specific heat capacity was calculated by comparing the heat flow of the measured samples with that of the standard sample. As shown in Supplementary Figure 8, a sharp peak is observed in pure water, followed by a significant decrease in the heat flow signal, indicating immediate water evaporation. However, the heat flow signals were differed in all the tested DSF samples, the peaks are much broader and lower than that of pure water.



**Supplementary Figure 9.** Raman Spectroscopy of Water Structure: **a.** Bulk water, **b.** PDA coated surface.

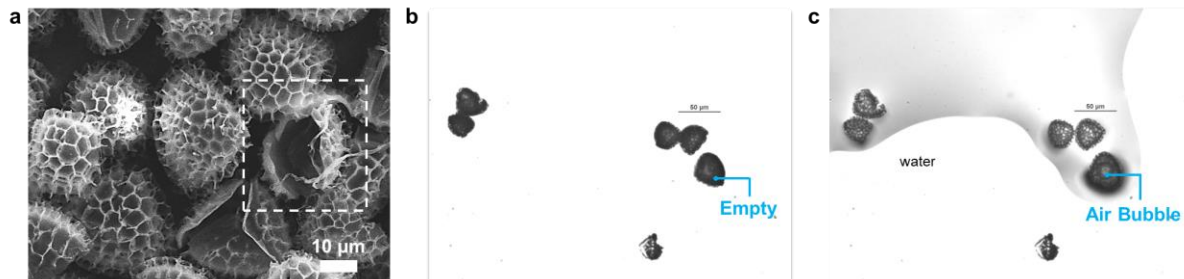
To elucidate the variation in hydrogen bonding between bulk water and the confined thin water layer on polymeric materials, we analyzed the Raman Spectra of hydrogen bond vibrations. Peaks at 3250, 3410, 3520, and 3630  $\text{cm}^{-1}$ , as displayed in Supplementary Figure 9, correspond to distinct water molecule types, aligning with reported literature. Compared to bulk water, the PDA surface exhibits a lower molar ratio of free water (FW), indicated by the peak at 3250  $\text{cm}^{-1}$ . The affinity of water with the surface increases the ratio of bound water (BW) at 3410  $\text{cm}^{-1}$ . Furthermore, the presence of intermediate water (IW), associated with peaks at 3520 and 3630  $\text{cm}^{-1}$ , is significantly higher on the PDA surface. These findings suggest that the confined water within the PDA-assembled photothermal layer disrupts the stable hydrogen bonding network found in bulk water, thereby enhancing water evaporation efficiency.



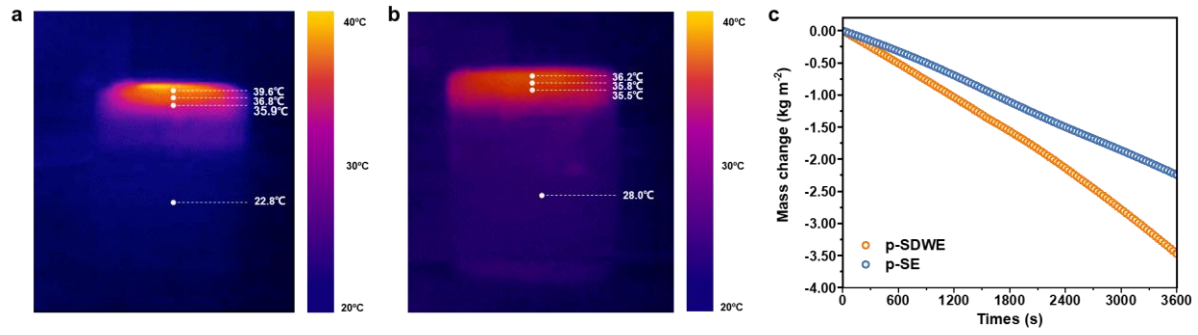


**Supplementary Figure 10.** The concentration of  $\text{Li}^+$  collected from evaporation by p-SDWE and natural evaporation.

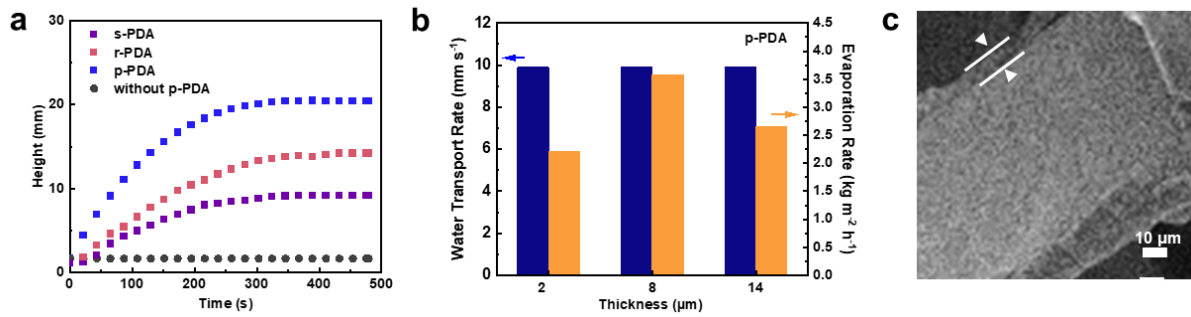
The collected water from our SDWE-based evaporation showed a  $\text{Li}^+$  concentration of approximately  $8.9 \text{ mg L}^{-1}$  after evaporated  $50 \text{ g L}^{-1}$  and  $56.3 \text{ mg L}^{-1}$  after evaporated  $100 \text{ g L}^{-1}$ , while water from natural evaporation methods showed a much lower  $\text{Li}^+$  concentration. These findings suggest that water evaporates in clusters from the SDWE, likely due to the strong solvation effect of  $\text{Li}^+$ . This conclusion, in conjunction with the Raman spectra analysis in Supplementary Figure 10 and the reduced latent heat detailed in Supplementary Table 1, supports the hypothesis that the existence of a thin water layer, enhanced by the surface characteristics of polymeric materials such as PDA, plays a significant role in boosting evaporation efficiency.



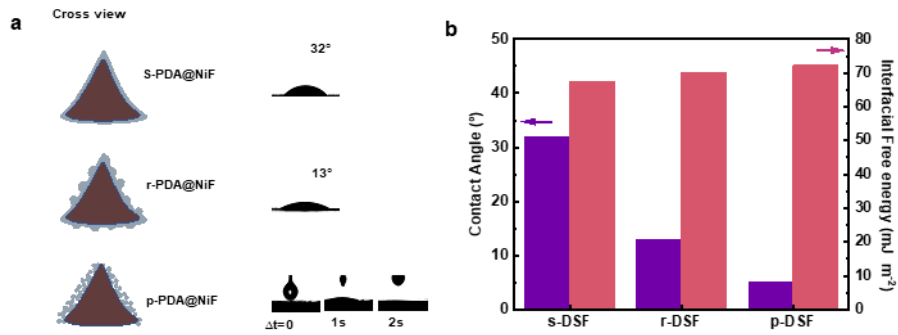
**Supplementary Figure 11.** a. SEM image of SEC, b. Optical microscopic image of SEC, c. Optical microscopic image of SEC immersed in water.



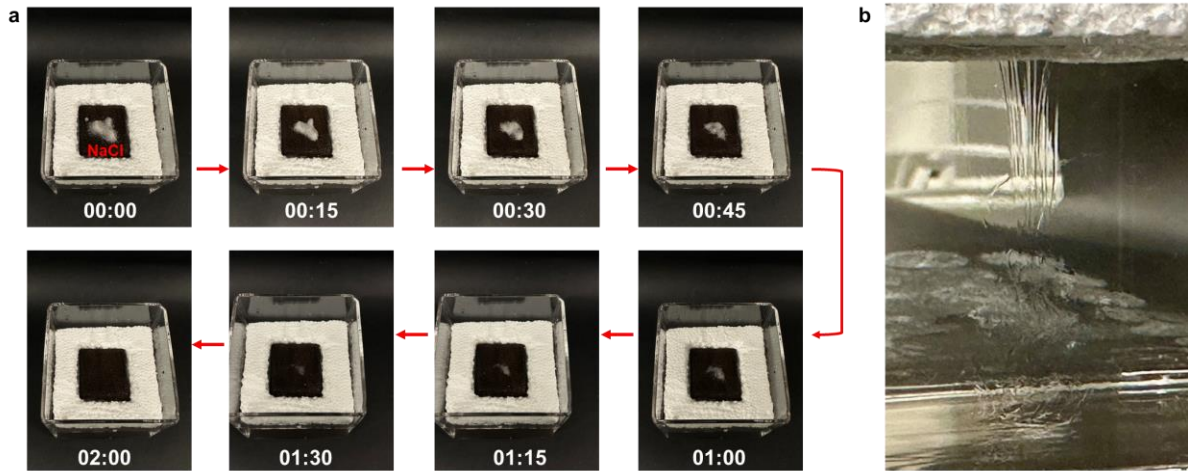
**Supplementary Figure 12.** The temperature distribution of the evaporator: a. p-SDWE, b. p-SE, as a reference sample, where the bottom without SEC layer, c. Mass change of evaporated water over time of the p-SDWE and the reference sample p-SE (without the SEC modification) under one sun solar illumination.



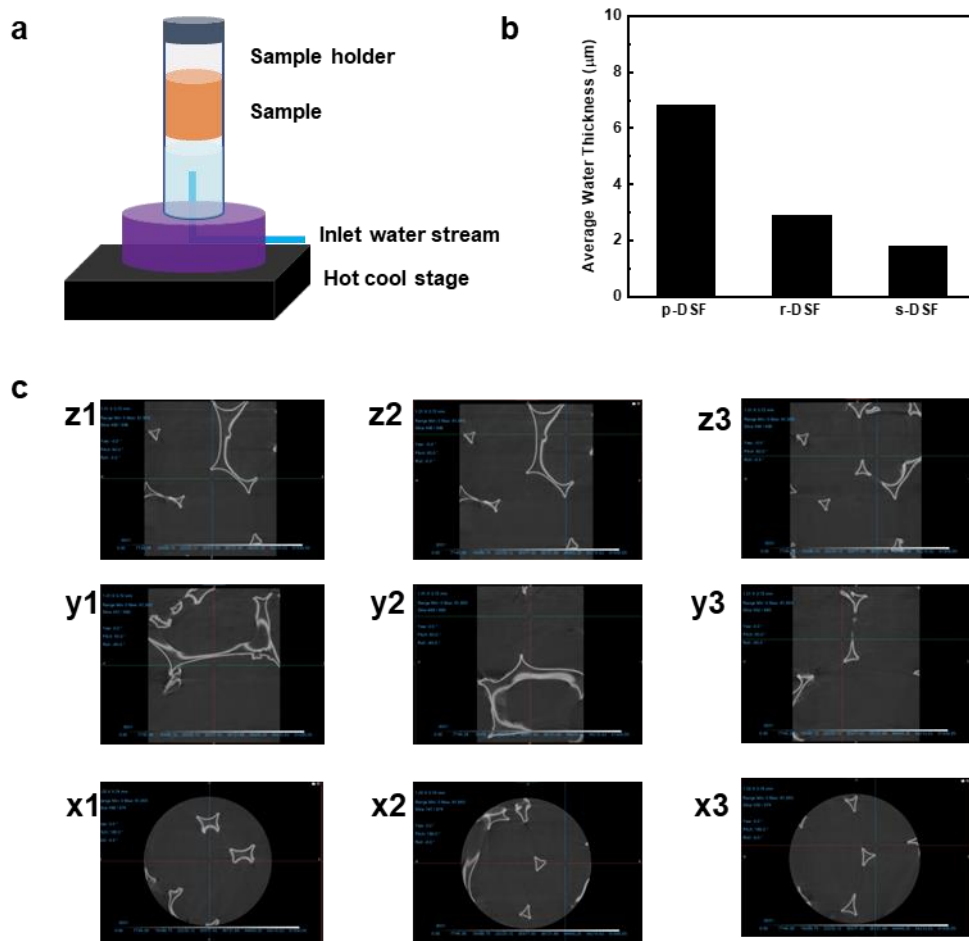
**Supplementary Figure 13. a.** Water transport rate along PDA layer varying different structures  
**b.** Illustration of the water transport rate and evaporation rate varying the thickness of PDA layer,  
**c.** The thickness of p-PDA layer on PN10-g-SEC



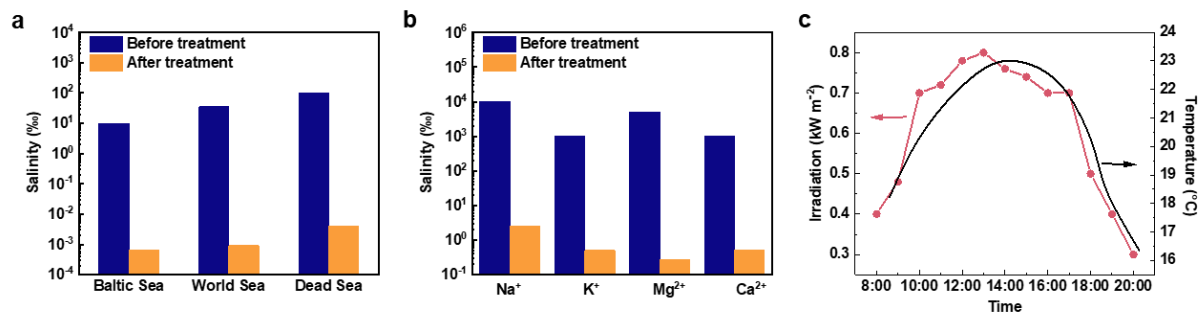
**Supplementary Figure 14. a.** Illustration of water affinity and its corresponding contact angles of PDA layer varying the structure, **b.** Contact angle and interfacial free energy of s-PDA, r-PDA and p-PDA layer.



**Supplementary Figure 15. a.** Illustration of the salt redissolving procedure with additional 1gram sodium chloride of p-SDWE under  $1\text{KW m}^{-2}$  illumination, **b.** Salt backflow driven by gradient and gravity

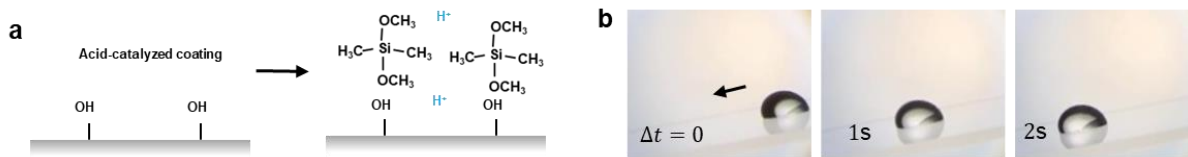


**Supplementary Figure 16. a.** Thin water layer characterization conducted by Micro-CT, **b.** Water thickness within s-SDWE, r-SDWE and p-SDWE, **c.** Cross view of water layer observed in p-SDWE



**Supplementary Figure 17. a.** The salinities of three artificial seawater samples before and after desalination using p-SDWE. **b.** Measured concentrations of four primary ions in an actual seawater sample before and after desalination, **c.** Solar radiation recorded over time on a sunny day from 08:00 to 20:00.





**Supplementary Figure 18. a.** Preparation of omniphobic liquid-like coating using acid-catalyzed method, **b.** Time-sequence images illustrating water slide on the omniphobic liquid-like coated glass substrate at a tilted angle of  $10^\circ$ .

**Supplementary Table 1.** Equivalent evaporation Enthalpy of Water and SDWEs

<b>enthalpy (J g<sup>-1</sup>)</b>	<b>s-DSF</b>	<b>r-DSF</b>	<b>p-DSF</b>	<b>water</b>
<b>DSC</b>	<b>1610</b>	<b>1780</b>	<b>1930</b>	<b>2420</b>
<b>vaporization test</b>	<b>1032</b>	<b>1050</b>	<b>1080</b>	

Supplementary Table 1 summarizes the total mass, mass of water, and calculated vaporization enthalpy of bulk water and our as-prepared samples. The measured enthalpy of water is 2420 J g<sup>-1</sup>, which closely to the theoretical value of 2450 J g<sup>-1</sup>, indicating the accuracy of our measurements. The vaporization enthalpy of water within the p-DSF was significantly lower than that of pure water. To determine the vaporization enthalpy, the evaporation enthalpy of water within DSF was measured. Specifically, water and DSF samples with the same surface area were placed together in a closed container along with a supersaturated potassium carbonate solution, creating a stabilized relative humidity (RH) under room temperature. The equivalent evaporation enthalpy ( $h_{eq}$ ) of water in SDWEs can be estimated by vaporizing water with the same power input ( $U_{in}$ ), using the equation (1):

$$U_{in} = h_{eq} m_0 = h_{vap} m_g$$

where  $h_{vap}$  was the evaporation enthalpy and  $m_0$  was the mass change of bulk water, while  $m_g$  referred to the mass change of the HNG sample. The evaporation rate of water and the corresponding calculated  $h_{eq}$ , where  $h_{eq}$  was 1930 J g<sup>-1</sup> for p-DSF, was found to be significantly reduced compared to bulk water.

**Supplementary Table 2.** A summary of reported evaporators, varying in structural design, fluidic flow principles, cycling performance, resistance to salt fouling, and evaporation rate.

Evaporator's structure	Fluidic flow	Salt removal methods	Cycling operation time	Evaporation rate (kg m <sup>-2</sup> h <sup>-1</sup> )	Solar-to-steam efficiency	Net interfacial evaporation rate (kg m <sup>-2</sup> h <sup>-1</sup> )	References
Metallic $\lambda$ -Ti3O5 nanoparticles-hydrogel	Concave cavity-induced flow	Salt reject (3.5wt%)	100 h	1.64 (2D evaporator) 6.09 (3D evaporator)	68.3% 95.9%	1.42 (2D evaporator) 4.31 (3D evaporator)	4
TEMPO-doped PDA-cellulose membrane	Unidirectional flow		1h*30	1.53	88.6%	1.362	5
2D Alumina sheet with aligned wicking tube	Vertical flow along the tube/microchannel	Open capillary tubes/reclean		1.26		0.62	6
Transformable 2D–3D	Directional	Light off wash		2.34 (2D evaporator) 4.35 (3D evaporator)		0.69 (2D evaporator) 1.34 (3D evaporator)	7
3D hydrogel	Unidirectional pump to the top surface		28 days	3.2	94%	2.9	8
3D sponge	Unidirectional flow with ultra-thin water supply	Day-night wash		3.2	91.6%	1.4	9
3D graphene foam with vertically oriented channel	Injection control water amount	Salt-resistant	100 h	2.40	99.4%	2.67	10
Multi-stage combined PV	Horizontal transport	Salt out-Salt crystallized at specific sites		2.45		1.77	11
Janus-interface solar-steam	Separated water and vapor			2.21	88%	1.95	12
Foam particles and aligned water channels	Directional flow	Salt resistant	100 h	2.25	136.7%	1.14	13
Wick-free confined water layer	Convective flow	Salt reject (20wt%)		1.36	91%	1.36	14
Contactless	Non contact	No salt (3.5wt%)		2.5	25%	2.5	15
Conical	Non-contact	Salt transport to specific area and recovery	40 h	1.25	89.9 %	1.25	16
Water evaporation surface and light absorption surface are physically separated	Directional flow	Salt crystallized at specific sites	288 h	2.42	94.3%	2.12	17
Asymmetric grooves and microcavity arrays	Marangoni flow	Salt crystallized at specific sites	9 h*10	2.63	96%	2.28	18
Thermo-responsive hydrogel	Unidirectional flow	Salt-free	10 h*7	4.145		N/A	19
Ammonia-Responsive +photocatalytic	Unidirectional flow		100 h	2.9 (2 sun)	97.3%	N/A	20
<b>Bilayer-structured thermo-responsive foam</b>	<b>Dynamic fluidic flow (thin water at evaporation transition to bulk water at salt self-washing)</b>	<b>Self washing</b>	<b>216 h</b>	<b>3.58</b>	<b>93.9%</b>	<b>3.13</b>	<b>This work</b>

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