

## Supplementary Material

### S.1 Non-linear and anisotropic constitutive model for the pulmonary artery

The isotropic and anisotropic strain energy functions are given by:

$$\bar{\Psi}_{iso} = \frac{c}{2}(\bar{I}_1 - 3), \quad (15)$$

$$\bar{\Psi}_{aniso} = \frac{k_1}{2k_2} \{ \exp[k_2(\bar{I}_4 - 1)^2] + \exp[k_2(\bar{I}_6 - 1)^2] - 2 \}, \quad (16)$$

where  $c$ ,  $k_1$  and  $k_2$  are material parameters.  $c$  is associated with the isotropic (ECM) response, while  $k_1$  and  $k_2$  are associated with the fiber (anisotropic) response. The Cauchy stress is given by

$$\bar{\boldsymbol{\sigma}} = c \operatorname{dev}(\bar{\mathbf{b}}) + 2\bar{\Psi}_4 \operatorname{dev}(\mathbf{a}_1 \otimes \mathbf{a}_1) + 2\bar{\Psi}_6 \operatorname{dev}(\mathbf{a}_2 \otimes \mathbf{a}_2), \quad (17)$$

$$\bar{\mathbf{b}} = \bar{\mathbf{F}}\bar{\mathbf{F}}^T, \quad (18)$$

$$\operatorname{dev}(\bullet) = (\bullet) - \frac{1}{3}[(\bullet) : \mathbf{I}]\mathbf{I}, \quad (19)$$

where  $\bar{\Psi}_4 = \partial \bar{\Psi}_{aniso} / \partial \bar{I}_4$  and  $\bar{\Psi}_6 = \partial \bar{\Psi}_{aniso} / \partial \bar{I}_6$  are the derivatives of the anisotropic strain energy function with respect to the invariants  $\bar{I}_4$  and  $\bar{I}_6$ .  $\bar{\mathbf{b}}$  is the left Cauchy-green tensor. The derivatives were computed to be

$$\frac{\partial \bar{\Psi}_{aniso}}{\partial \bar{I}_4} = k_1(\bar{I}_4 - 1) \exp[k_2(\bar{I}_4 - 1)^2], \quad (20)$$

$$\frac{\partial \bar{\Psi}_{aniso}}{\partial \bar{I}_6} = k_1(\bar{I}_6 - 1) \exp[k_2(\bar{I}_6 - 1)^2]. \quad (21)$$

For an ideal case, assume the following isochoric deformation vector with no shear components

$$\bar{\mathbf{F}} = \begin{bmatrix} \lambda_r & 0 & 0 \\ 0 & \lambda_\theta & 0 \\ 0 & 0 & \lambda_z \end{bmatrix}, \quad \text{where } \lambda_r \lambda_\theta \lambda_z = 1, \quad (22)$$

$$\bar{\mathbf{b}} = \bar{\mathbf{C}} = \begin{bmatrix} \lambda_r^2 & 0 & 0 \\ 0 & \lambda_\theta^2 & 0 \\ 0 & 0 & \lambda_z^2 \end{bmatrix}, \quad (23)$$

$$\operatorname{dev}(\bar{\mathbf{b}}) = \begin{bmatrix} \frac{2}{3}\lambda_r^2 - \frac{1}{3}(\lambda_\theta^2 + \lambda_z^2) & 0 & 0 \\ 0 & \frac{2}{3}\lambda_\theta^2 - \frac{1}{3}(\lambda_r^2 + \lambda_z^2) & 0 \\ 0 & 0 & \frac{2}{3}\lambda_z^2 - \frac{1}{3}(\lambda_r^2 + \lambda_\theta^2) \end{bmatrix}. \quad (24)$$

Hence, the isotropic stress is given by

$$\bar{\sigma}_{iso} = c \begin{bmatrix} \frac{2}{3}\lambda_r^2 - \frac{1}{3}(\lambda_\theta^2 + \lambda_z^2) & 0 & 0 \\ 0 & \frac{2}{3}\lambda_\theta^2 - \frac{1}{3}(\lambda_r^2 + \lambda_z^2) & 0 \\ 0 & 0 & \frac{2}{3}\lambda_z^2 - \frac{1}{3}(\lambda_r^2 + \lambda_\theta^2) \end{bmatrix}. \quad (25)$$

Considering the effects of the fibers,  $\mathbf{a}_1 = \bar{\mathbf{F}}\mathbf{a}_{01}$ , where  $a_{01}$  and  $a_{02}$  are given by the fiber angle  $\beta$ .

$$a_{01} = \begin{bmatrix} 0 \\ \cos \beta \\ \sin \beta \end{bmatrix}, \quad a_{02} = \begin{bmatrix} 0 \\ \cos \beta \\ -\sin \beta \end{bmatrix}, \quad (26)$$

$$\mathbf{a}_1 = \begin{bmatrix} \lambda_r & 0 & 0 \\ 0 & \lambda_\theta & 0 \\ 0 & 0 & \lambda_z \end{bmatrix} \begin{bmatrix} 0 \\ \cos \beta \\ \sin \beta \end{bmatrix}, \quad \mathbf{a}_2 = \begin{bmatrix} \lambda_r & 0 & 0 \\ 0 & \lambda_\theta & 0 \\ 0 & 0 & \lambda_z \end{bmatrix} \begin{bmatrix} 0 \\ \cos \beta \\ -\sin \beta \end{bmatrix}, \quad (27)$$

$$\mathbf{a}_1 = \begin{bmatrix} 0 \\ \lambda_\theta \cos \beta \\ \lambda_z \sin \beta \end{bmatrix}, \quad \mathbf{a}_2 = \begin{bmatrix} 0 \\ \lambda_\theta \cos \beta \\ -\lambda_z \sin \beta \end{bmatrix}. \quad (28)$$

$$(29)$$

Now converting  $\mathbf{a}_1$  and  $\mathbf{a}_2$  into matrices and determining the deviatoric part

$$\mathbf{a}_1 \otimes \mathbf{a}_1 = \begin{bmatrix} 0 & 0 & 0 \\ 0 & (\lambda_\theta \cos \beta)^2 & \lambda_\theta \lambda_z \cos \beta \sin \beta \\ 0 & \lambda_\theta \lambda_z \cos \beta \sin \beta & (\lambda_z \sin \beta)^2 \end{bmatrix}, \quad (30)$$

$$dev(\mathbf{a}_1 \otimes \mathbf{a}_1) = \begin{bmatrix} -\frac{1}{3} \left( (\lambda_\theta \cos \beta)^2 + (\lambda_z \sin \beta)^2 \right) & 0 & 0 \\ 0 & \frac{2}{3} (\lambda_\theta \cos \beta)^2 - \frac{1}{3} (\lambda_z \sin \beta)^2 & \lambda_\theta \lambda_z \cos \beta \sin \beta \\ 0 & \lambda_\theta \lambda_z \cos \beta \sin \beta & \frac{2}{3} (\lambda_z \sin \beta)^2 - \frac{1}{3} (\lambda_\theta \cos \beta)^2 \end{bmatrix}, \quad (31)$$

$$\mathbf{a}_2 \otimes \mathbf{a}_2 = \begin{bmatrix} 0 & 0 & 0 \\ 0 & (\lambda_\theta \cos \beta)^2 & -\lambda_\theta \lambda_z \cos \beta \sin \beta \\ 0 & -\lambda_\theta \lambda_z \cos \beta \sin \beta & (\lambda_z \sin \beta)^2 \end{bmatrix}, \quad (32)$$

$$dev(\mathbf{a}_2 \otimes \mathbf{a}_2) = \begin{bmatrix} -\frac{1}{3} \left( (\lambda_\theta \cos \beta)^2 + (\lambda_z \sin \beta)^2 \right) & 0 & 0 \\ 0 & \frac{2}{3} (\lambda_\theta \cos \beta)^2 - \frac{1}{3} (\lambda_z \sin \beta)^2 & -\lambda_\theta \lambda_z \cos \beta \sin \beta \\ 0 & -\lambda_\theta \lambda_z \cos \beta \sin \beta & \frac{2}{3} (\lambda_z \sin \beta)^2 - \frac{1}{3} (\lambda_\theta \cos \beta)^2 \end{bmatrix}. \quad (33)$$

Now computing the stress  $\sigma_{\theta\theta}$

$$\begin{aligned} \bar{\sigma}_{rr} = & c \left[ \frac{2}{3} \lambda_r^2 - \frac{1}{3} (\lambda_\theta^2 + \lambda_z^2) \right] - \frac{2k_1(\bar{I}_4 - 1)}{3} \exp[k_2(\bar{I}_4 - 1)^2] \left[ (\lambda_\theta \cos \beta)^2 + (\lambda_z \sin \beta)^2 \right] \\ & - \frac{2k_1(\bar{I}_6 - 1)}{3} \exp[k_2(\bar{I}_6 - 1)^2] \left[ (\lambda_\theta \cos \beta)^2 + (\lambda_z \sin \beta)^2 \right], \end{aligned} \quad (34)$$

$$\begin{aligned} \bar{\sigma}_{\theta\theta} = & c \left[ \frac{2}{3} \lambda_\theta^2 - \frac{1}{3} (\lambda_r^2 + \lambda_z^2) \right] + \frac{2k_1(\bar{I}_4 - 1)}{3} \exp[k_2(\bar{I}_4 - 1)^2] \left[ 2(\lambda_\theta \cos \beta)^2 - (\lambda_z \sin \beta)^2 \right] \\ & + \frac{2k_1(\bar{I}_6 - 1)}{3} \exp[k_2(\bar{I}_6 - 1)^2] \left[ 2(\lambda_\theta \cos \beta)^2 - (\lambda_z \sin \beta)^2 \right], \end{aligned} \quad (35)$$

$$\begin{aligned} \bar{\sigma}_{zz} = & c \left[ \frac{2}{3} \lambda_z^2 - \frac{1}{3} (\lambda_r^2 + \lambda_\theta^2) \right] + \frac{2k_1(\bar{I}_4 - 1)}{3} \exp[k_2(\bar{I}_4 - 1)^2] \left[ 2(\lambda_z \sin \beta)^2 - (\lambda_\theta \cos \beta)^2 \right] \\ & + \frac{2k_1(\bar{I}_6 - 1)}{3} \exp[k_2(\bar{I}_6 - 1)^2] \left[ 2(\lambda_z \sin \beta)^2 - (\lambda_\theta \cos \beta)^2 \right]. \end{aligned} \quad (36)$$

Now computing the fiber-related invariants

$$\bar{I}_4 = \bar{\mathbf{C}} : (a_{01} \otimes a_{01}), \quad \text{and} \quad \bar{I}_6 = \bar{\mathbf{C}} : (a_{02} \otimes a_{02}), \quad (37)$$

$$\implies \bar{I}_4 = \lambda_\theta^2 \cos^2 \beta + \lambda_z^2 \sin^2 \beta, \quad \text{and} \quad \bar{I}_6 = \lambda_\theta^2 \cos^2 \beta + \lambda_z^2 \sin^2 \beta. \quad (38)$$

The final stress values are

$$\begin{aligned} \bar{\sigma}_{rr} = & c \left[ \frac{2}{3} \lambda_r^2 - \frac{1}{3} (\lambda_\theta^2 + \lambda_z^2) \right] \\ & - \frac{4k_1}{3} \exp[k_2 (\lambda_\theta^2 \cos^2 \beta + \lambda_z^2 \sin^2 \beta - 1)^2] \left[ (\lambda_\theta \cos \beta)^2 + (\lambda_z \sin \beta)^2 \right] \left[ \lambda_\theta^2 \cos^2 \beta + \lambda_z^2 \sin^2 \beta - 1 \right], \end{aligned} \quad (39)$$

$$\begin{aligned} \bar{\sigma}_{\theta\theta} = & c \left[ \frac{2}{3} \lambda_\theta^2 - \frac{1}{3} (\lambda_r^2 + \lambda_z^2) \right] \\ & + \frac{4k_1}{3} \exp[k_2 (\lambda_\theta^2 \cos^2 \beta + \lambda_z^2 \sin^2 \beta - 1)^2] \left[ 2(\lambda_\theta \cos \beta)^2 - (\lambda_z \sin \beta)^2 \right] \left[ \lambda_\theta^2 \cos^2 \beta + \lambda_z^2 \sin^2 \beta - 1 \right], \end{aligned} \quad (40)$$

$$\begin{aligned} \bar{\sigma}_{zz} = & c \left[ \frac{2}{3} \lambda_z^2 - \frac{1}{3} (\lambda_r^2 + \lambda_\theta^2) \right] \\ & + \frac{4k_1}{3} \exp[k_2 (\lambda_\theta^2 \cos^2 \beta + \lambda_z^2 \sin^2 \beta - 1)^2] \left[ 2(\lambda_z \sin \beta)^2 - (\lambda_\theta \cos \beta)^2 \right] \left[ \lambda_\theta^2 \cos^2 \beta + \lambda_z^2 \sin^2 \beta - 1 \right]. \end{aligned} \quad (41)$$

We know that  $\sigma = \sigma_{vol} + \bar{\sigma}$ , where  $\sigma_{vol} = p\mathbf{I}$  is the hydrostatic stress and  $p$  is the hydrostatic pressure.

## S.2 Fluid pressure during inflation of vessel

For pure inflation of a cylinder, the variables in the deformed configuration are given by  $r = r(R)$ ,  $\theta = \Theta$ , and  $z = Z$ . Here  $R \in [R_i, R_o]$ ,  $\Theta \in [0, 2\pi]$  and let  $R_o - R_i = h$  be the thickness. Hence, the deformation gradient and Cauchy green tensors are given by

$$\bar{\mathbf{F}} = \begin{bmatrix} \frac{R}{r} & 0 & 0 \\ 0 & \frac{r}{R} & 0 \\ 0 & 0 & 1 \end{bmatrix}, \quad (42)$$

$$\bar{\mathbf{b}} = \begin{bmatrix} \left(\frac{R}{r}\right)^2 & 0 & 0 \\ 0 & \left(\frac{r}{R}\right)^2 & 0 \\ 0 & 0 & 1 \end{bmatrix}. \quad (43)$$

We can compute hydrostatic pressure using the equilibrium equation  $\nabla \sigma = 0$ . In cylindrical coordinates

$$\frac{\partial \sigma_{rr}}{\partial r} + \frac{\sigma_{rr} - \sigma_{\theta\theta}}{r} = 0. \quad (44)$$

We use this to get the value of internal pressure as

$$P_i = \int_{r_i}^{r_o} \frac{\sigma_{\theta\theta} - \sigma_{rr}}{r} dr. \quad (45)$$

We can substitute  $\sigma_{rr} - \sigma_{\theta\theta} = \bar{\sigma}_{rr} - \bar{\sigma}_{\theta\theta}$ , because  $\sigma_{rr} = \bar{\sigma}_{rr} + p$ , and  $\sigma_{\theta\theta} = \bar{\sigma}_{\theta\theta} + p$ , where  $p$  is the hydrostatic pressure. Hence

$$P_i = \int_{r_i}^{r_o} \frac{\bar{\sigma}_{\theta\theta} - \bar{\sigma}_{rr}}{r} dr, \quad (46)$$

Computing the RHS

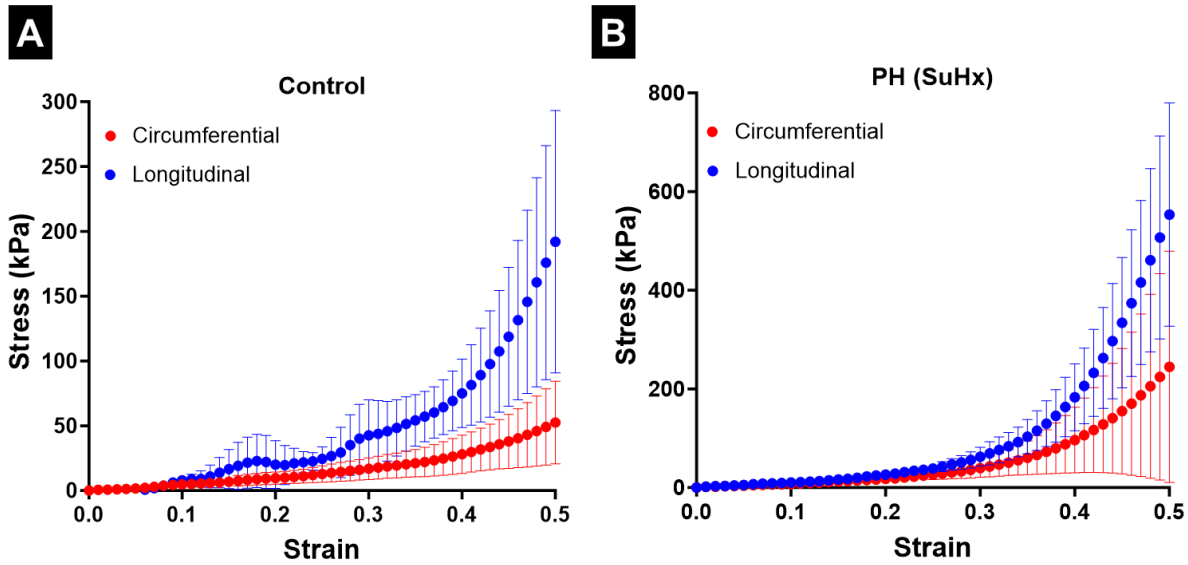
$$\begin{aligned} \bar{\sigma}_{\theta\theta} - \bar{\sigma}_{rr} &= c [\lambda_{\theta}^2 - \lambda_r^2] \\ &+ \frac{4k_1}{3} \exp[k_2 ([\lambda_{\theta}^2 \cos^2 \beta + \lambda_z^2 \sin^2 \beta] - 1)^2] [3 (\lambda_{\theta} \cos \beta)^2] [\lambda_{\theta}^2 \cos^2 \beta + \lambda_z^2 \sin^2 \beta - 1]. \end{aligned} \quad (47)$$

Substituting the values for  $\lambda_r$ ,  $\lambda_{\theta}$ ,  $\lambda_z$ , we get

$$\begin{aligned} \bar{\sigma}_{\theta\theta} - \bar{\sigma}_{rr} &= c \left[ \left(\frac{r}{R}\right)^2 - \left(\frac{R}{r}\right)^2 \right] \\ &+ \frac{4k_1}{3} \exp \left[ k_2 \left( \left[ \left(\frac{r}{R}\right)^2 \cos^2 \beta + \sin^2 \beta \right] - 1 \right)^2 \right] \left[ 3 \left( \left(\frac{r}{R}\right) \cos \beta \right)^2 \right] \left[ \left(\frac{r}{R}\right)^2 \cos^2 \beta + \sin^2 \beta - 1 \right]. \end{aligned} \quad (48)$$

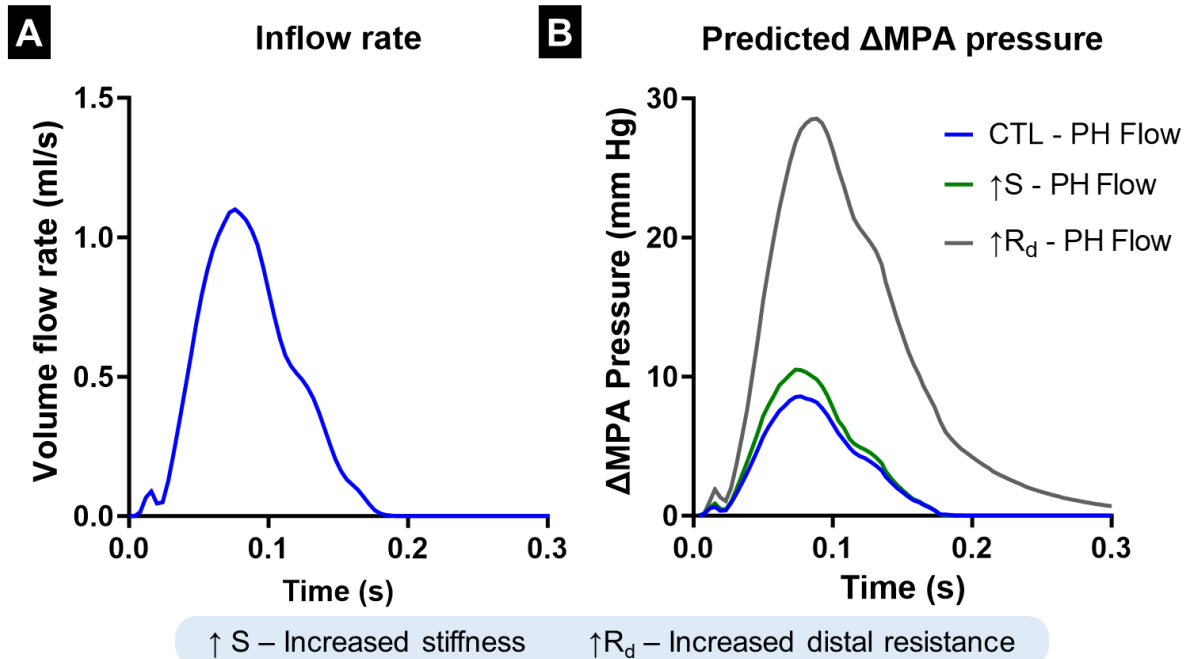
We can compute  $P_i$  by integrating Eq. S46 over the thickness of the vessel.

### S.3 Uniaxial testing behavior



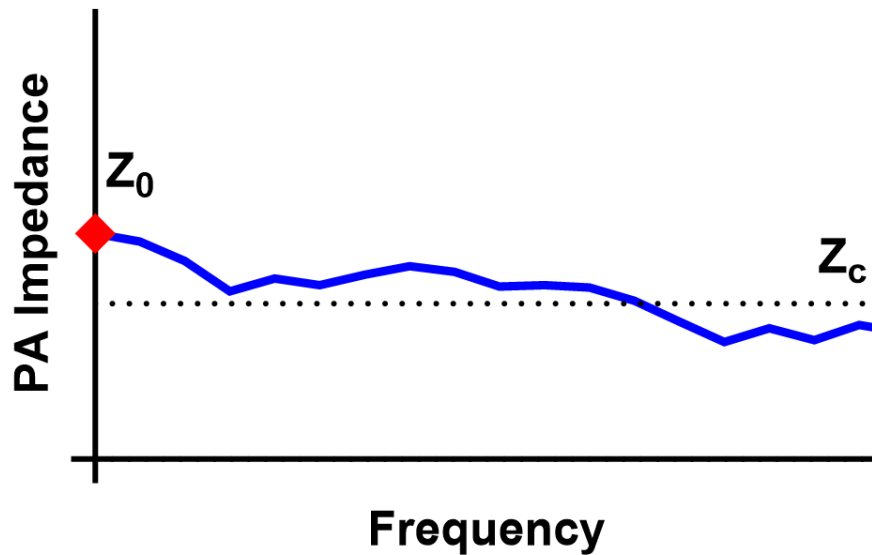
**Figure S1.** Uniaxial testing of MPA tissue specimens from (A) control rats, (B) PH rats. CTL: n=6, PH: n=6.

### S.4 Effect of remodeling parameters on pressure waveform



**Figure S2.** Effect of individual vascular remodeling events on the pressure waveform. (A) Volume flow rate at the inlet, (B) Pressure waveform for increased vessel stiffness and distal resistance. (C) Volume flow rate at the inlet with an increased period of zero flow after the pulse, (B) Pressure waveform increased distal resistance to study the relaxation behavior.

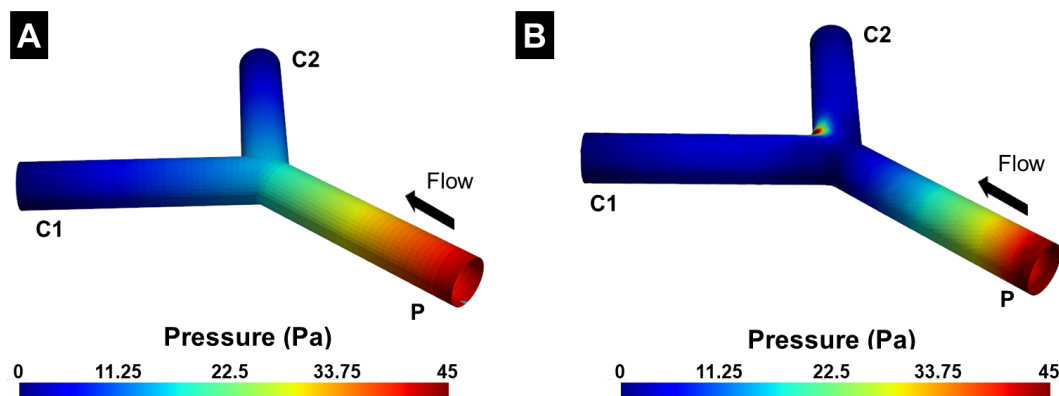
## S.5 Variation of PA impedance with frequency



**Figure S3.** Schematic describing the variation of impedance with frequency and the estimation of 0Hz- and characteristic impedance.

## S.6 Validation against 3D FSI

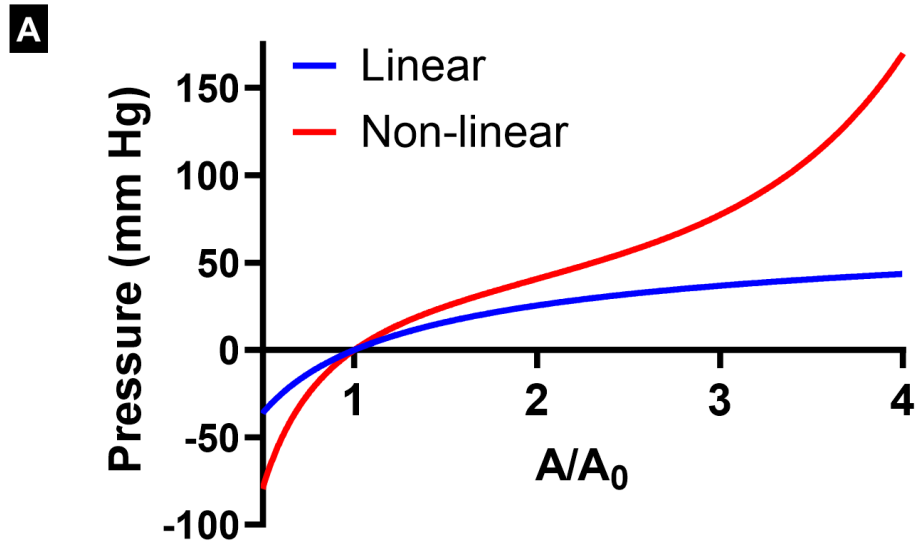
For validation, we have compared the 1D FSI framework presented in this study against a 3D FSI framework. Here we simulated a constant fluid flow rate through a Y-branch, where the length and radius of each vessel are 50mm and 5mm respectively. The thickness is set to be 20% of the radius (1mm). The flow rate is set to 30ml/s. For the vessel properties, the stiffness of fiber components was set to 0, and the shear modulus ( $c$ ) was set to 60kPa. To perform the equivalent simulation in 3D, we performed the simulation in ANSYS workbench, coupling the transient structural and Fluent modules. Here, the displacement of the inlet and outlet faces was constrained such that only radial and circumferential displacement was allowed. A single additional point on the inlet was completely fixed to ensure stable simulations.



**Figure S4.** (A) The pressure predicted by the 1D simulation mapped onto a representative geometry, and (B) Pressure on the wall of the fluid domain. **P**: Parent vessel; **C1**, **C2**: child vessels.

The results indicated good agreement between the 1D and 3D simulations. The pressure at the inlet in the 1D simulation was 42.8 Pa. Correspondingly, the pressure on the fluid domain wall from the 3D simulation was 44.7Pa. The difference between the predicted values is likely due to the idealized bifurcation assumed by the 1D FSI framework.

### S.7 Effect of non-linear material model



**Figure S5.** Variation of pressure with lumen area. Lumen area was normalized by the original lumen area