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2 **Supplementary Information for**
3 **Optimal transport and control of active drops**

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15 1. Controllability and finite dimensional optimal control

16 We briefly outline the relevant control theoretic notions that allow us to formulate and solve the ODE formulation of the
17 optimal transport problem. Consider a general dynamical system evolving on a smooth n -dimensional manifold \mathcal{M} (the state
18 space), of the form

$$19 \quad \dot{y}(t) = f_0(y(t)) + \sum_{i=1}^m u_i(t) f_i(y(t)), \quad [S1]$$

20 where $u_i(t)$, $i = 1, \dots, m$ are the control variables, $f_0(y)$ is a drift vector field, and $V(y) = \{f_i(y) \in T_y\mathcal{M}, i = 1, \dots, m\}$ is
21 the set of nonlinear control vector fields in the tangent space ($T_y\mathcal{M}$) at a point $y \in \mathcal{M}$. This system is said to be *drift-free*
22 if $f_0(y) = 0$, and *underactuated* if there are fewer controls than the dimensionality of the manifold, i.e., $m < \dim(\mathcal{M}) = n$.
23 When $m < n$, let $\mathcal{V} = \cup_{y \in \mathcal{M}} \text{span}(V(y)) \subset T\mathcal{M}$ be the m -dimensional restricted sub-bundle within the tangent bundle
24 $T\mathcal{M} = \cup_{y \in \mathcal{M}} T_y\mathcal{M}$. In the following, we limit our attention to drift-free systems and state sufficient conditions for the existence
25 of controls in this setting.

Definition 1.1 (Controllability) Consider the drift-free dynamics

$$26 \quad \dot{y}(t) = \sum_{i=1}^m u_i(t) f_i(y(t)) \quad [S2]$$

27 evolving on a smooth n -dimensional manifold \mathcal{M} . This system is said to be *controllable* if there exist controls u_i that steer the
28 state from any initial configuration $y_0 \in \mathcal{M}$ to any final configuration $y_1 \in \mathcal{M}$.

29 The definition of controllability stated here does not specify the duration required to achieve the state transfer. For the
30 special case of linear dynamical systems taking the form $\dot{y} = Ay + Bu$, controllability implies finite time controllability. In fact
31 for this dynamics, if a prescribed state transfer is possible, it is possible in arbitrarily small time (1). However, more general
32 systems such as S2 may be controllable, and yet not finite-time controllable.

33 For nonlinear dynamical systems such as in Eq. S2, the non-commutativity of the control vector fields (measured by the Lie
34 bracket) plays a crucial role in determining the controllability of Eq. S1.

Definition 1.2 (Lie Bracket) For two vector fields $f_i(y)$, $f_j(y) \in V(y)$, their Lie bracket is defined as

$$35 \quad [f_i(y), f_j(y)] = \nabla_y f_j(y) f_i(y) - \nabla_y f_i(y) f_j(y). \quad [S3]$$

36 The Lie bracket dictates the tangent direction along which the dynamical system is steered under an infinitesimal cyclic
37 actuation of the respective modulating controls u_i and u_j . The Lie algebra generated by the vector fields in $V(y)$, denoted by
38 $\{f_i(y), 1 \leq i \leq m\}_{L.A.}$, is constructed by equipping the vector space $V(y)$ with the Lie bracket operation (Eq. S3).

39 The theorem of Chow-Rashevsky (2) then provides the sufficient condition for the dynamics to be controllable.

40 **Theorem 1.1 (Chow-Rashevsky theorem)** Consider the dynamical system (S2) defined on a smooth manifold \mathcal{M} of
41 dimension n . This system is controllable if for all $y \in \mathcal{M}$, there exist n linearly independent vector fields in the Lie algebra
42 $\{f_i(y), 1 \leq i \leq m\}_{L.A.}$ generated by $\{f_i\}_{i=1}^m$, that span the tangent space $T_y\mathcal{M}$.

43 Theorem 1.1 provides a test for the global notion of controllability that depends on a local quantity, the Lie bracket. For
44 dynamics with drift such as in Eq. S1, the ability to steer in any direction in the neighbourhood of a point in \mathcal{M} is restricted
45 by the drift vector field. Hence, in such cases, local notions of accessibility and controllability need to be considered, but we do
46 not discuss them here for simplicity (see Ref. (2) for a pedagogical introduction).

47 **A. Optimal control problem formulation.** Consider again a drift-free system (Eq. S2) on the smooth manifold \mathcal{M} ($\dim(\mathcal{M}) = n$):

$$48 \quad \dot{y}(t) = \sum_{i=1}^m u_i(t) f_i(y(t)) \equiv \mathbf{\Omega}(y(t)) \mathbf{u}(t), \quad [S4]$$

49 where $\mathbf{\Omega}(y) = [f_1(y) \cdots f_m(y)]$, and $\mathbf{u}(t) = [u_1(t) \cdots u_m(t)]^T$. Suppose that our goal is to steer the state from $y(0)$ to $y(T)$
50 in finite time T , along a trajectory governed by Eq. S2, while minimizing the cost functional \mathcal{C}

$$51 \quad \mathcal{C} = \int_0^T dt \mathcal{L}(t, y, \mathbf{u}). \quad [S5]$$

52 In general, the Lagrangian \mathcal{L} can depend on the state (y), the controls (\mathbf{u}), but also on time (t). In the simplest setting, the
53 Lagrangian involves a quadratic form $\mathcal{L} = \mathbf{u}^T \mathbf{M}(y) \mathbf{u}$ with a state dependent positive semi-definite matrix $\mathbf{M}(y)$. Furthermore,
54 if the controls entering \mathcal{L} can be related to and eliminated in favour of the state dynamics (using Eq. S4), then the cost function
55 takes the form

$$56 \quad \mathcal{C} = \int_0^T dt \dot{y}(t)^T g(y(t)) \dot{y}(t), \quad [S6]$$

57 where $g(y)$ is a symmetric matrix, which when nondegenerate and positive definite, endows the manifold \mathcal{M} with a metric
58 tensor. In the case when $m = n$ and $\mathbf{\Omega}(y)$ has full rank (so is invertible), $g(y) = [\mathbf{\Omega}(y)^{-1}]^T \mathbf{M}(y) \mathbf{\Omega}(y)$ is an $n \times n$ symmetric
59 matrix that provides a Riemannian metric (when nondegenerate) on \mathcal{M} . For $m < n$, the dynamics in Eq. S4 may still admit a
60 rewriting of the cost as Eq. S6, with $g(y)$ now being an $m \times m$ symmetric positive definite matrix that defines a *sub-Riemannian*
61 metric on the distribution $\mathcal{V} \subset T\mathcal{M}$ instead (3, 4). In either case, the problem of finding optimal solutions for the steering the
62 state then reduces to a geometric problem, one of finding a minimizing geodesic connecting the desired initial and final states,
63 with the appropriate metric $g(y)$ induced by the Lagrangian.

64 If Eq. S4 is controllable, then there exists a solution to the state transfer problem. With this assumption, we can use
65 variational calculus to write down the first order necessary conditions for optimality of the controls, which we discuss below.

66 **B. Pontryagin Maximum Principle.** For a controllable dynamical system of the form in Eq. S4, Pontryagin's Maximum Principle
67 (5, 6) prescribes the first order necessary conditions for optimality.

Theorem 1.2 (Pontryagin's Maximum Principle (PMP)) *Suppose there exist optimal controls $\mathbf{u}^*(t) = [u_1^*(t) \cdots u_m^*(t)]^T$ that minimize the cost \mathcal{C} in Eq. S5 along trajectories satisfying Eq. S4, and the corresponding optimal state trajectory is denoted as $\mathbf{y}^*(t)$. Then, there exists a costate trajectory $\mathbf{p}(t)$ such that*

$$\dot{\mathbf{y}}^* = \partial_{\mathbf{p}} H(t, \mathbf{y}^*, \mathbf{p}, \mathbf{u}^*), \quad [\text{S7}]$$

$$\dot{\mathbf{p}} = -\partial_{\mathbf{y}^*} H(t, \mathbf{y}^*, \mathbf{p}, \mathbf{u}^*), \quad [\text{S8}]$$

68 where the Hamiltonian,

$$H(t, \mathbf{y}, \mathbf{p}, \mathbf{u}^*) = \max_{\mathbf{u}} \mathcal{H}(t, \mathbf{y}, \mathbf{p}, \mathbf{u}), \quad [\text{S9}]$$

69 is defined as the maximum of the pre-Hamiltonian \mathcal{H} over the controls. For the cost in Eq. S5, the pre-Hamiltonian is defined as

$$\mathcal{H}(t, \mathbf{y}, \mathbf{p}, \mathbf{u}) = \mathbf{p}^T \mathbf{\Omega}(y) \mathbf{u} - \mathcal{L}(t, \mathbf{y}, \mathbf{u}). \quad [\text{S10}]$$

72 If the system (Eq. S2) is controllable with $m < n$, and the quadratic cost \mathcal{C} can be written in terms of a positive definite
73 metric g (Eq. S6), then we have an optimal control problem with a sub-Riemannian metric endowed by the cost on the
74 m -dimensional distribution \mathcal{V} spanned by the set of control vector fields. This is called a sub-Riemannian optimal control
75 problem (4).

76 2. Reduced order dynamics and ODE drop control

77 Here we provide the details of the Galerkin projection calculation to obtain the finite dimensional ODE model of drop dynamics
78 from the continuum equations. We also derive the optimal control solution for the two parameter (X, R) description of drop
79 transport, extend the parametrization to account for drop shape and discuss the associated optimal control problem. In
80 addition to providing a finite dimensional representation, this formulation is also ripe for the direct application of results from
81 optimal control theory to check feasibility as well as provide, to the extent possible, analytical expressions for the transport
82 plan.

83 **A. Two parameter model reduction.** The continuum dynamics of the drop is given by a continuity equation for the drop height
84 $h(x, t)$ ($\int dx h = 1$)

$$\partial_t h + \partial_x q = 0, \quad q = \frac{h^3}{3\eta} \partial_x (\zeta h + \gamma \partial_x^2 h), \quad [\text{S11}]$$

86 driven by spatiotemporally varying activity $\zeta(x, t)$ and a constant surface tension γ . As described in the main text, in the
87 absence of activity ($\zeta = 0$), the flux vanishes when $\partial_x^3 h = 0$, i.e., $h(x, t)$ is parabolic,

$$h(x, t) = \frac{6}{R(t)^3} \left[\frac{R(t)^2}{4} - (x - X(t))^2 \right], \quad [\text{S12}]$$

89 and is parametrized by its position $X(t) = \int dx x h(x, t)$ and its size $R(t) = \sqrt{20\Delta(t)}$ (which is related as to the variance
90 $\Delta(t) = \int dx (x - X(t))^2 h(x, t)$). In order to obtain the effective dynamics of the drop in terms of \dot{X} and \dot{R} , we take moments
91 of Eq. S11 to get

$$\dot{X}(t) = \int_{X(t)-R(t)/2}^{X(t)+R(t)/2} dx q(x, t), \quad \dot{R}(t) = \int_{X(t)-R(t)/2}^{X(t)+R(t)/2} dx (x - X(t)) q(x, t). \quad [\text{S13}]$$

93 The limits $X(t) \pm R(t)/2$ are simply the ends of the drop, beyond which $h = 0$. This provides a closed system of equations to
94 describe drop dynamics on the parametrized manifold invariant to surface tension flows, upon which the controlled dynamics
95 steered by activity occurs.

96 **B. Two dimensional ODE optimal control.** Upon evaluating Eq. S13 along with the activity parametrization (Eq. 4, main text),
 97 we obtain the drop position and size dynamics to be

$$98 \quad \dot{X}(t) = \frac{18\Delta\zeta(t)}{35\eta R(t)^4}, \quad \dot{R}(t) = -\frac{24\zeta_0(t)}{7\eta R(t)^4}. \quad [S14]$$

99 As expected, the mean and gradient components of the active stress independently control the drop size and position respectively.
 100 This, together with non-vanishing control vector fields for finite R , trivially ensure controllability. For simplicity, we shall
 101 consider the fixed end point problem, in which case the cost $\mathcal{C} = \mathcal{W}$ is simply the net dissipation, $\mathcal{W} = \int_0^T dt \mathcal{L}$, where the
 102 Lagrangian is $\mathcal{L} = (1/\eta R^6)[(72/35)\zeta_0^2 + (54/77)\Delta\zeta^2]$. The Lagrangian is simply quadratic in the controls $(\zeta, \Delta\zeta)$, and has
 103 a strong size dependence due to the nonlinear dependence of the dissipation and friction on the height of the drop. Upon
 104 denoting the state vector as $\Psi(t) = [X(t) R(t)]^T$ and the costate (Lagrange multipliers) as $p(t) = [p_X(t) p_R(t)]^T$ to enforce
 105 the dynamical constraints in Eq. S14, a necessary condition for optimality is given by PMP. We then maximize the control
 106 Hamiltonian

$$107 \quad \mathcal{H}(\Psi, p, \zeta_0, \Delta\zeta) = p_X \frac{18\Delta\zeta}{35\eta R^4} - p_R \frac{24\zeta_0}{7\eta R^4} - \mathcal{L}, \quad [S15]$$

108 over the controls to get $\zeta_0^* = -5p_R R^2/6$ and $\Delta\zeta^* = 11p_X R^2/30$ as the maximizers. When plugged back into Eq. S15, this
 109 gives the conserved Hamiltonian (as it doesn't depend explicitly on time)

$$110 \quad H(\Psi, p) \equiv \mathcal{H}(\Psi, p, \zeta_0^*, \Delta\zeta^*) = \frac{1}{7\eta R^2} \left(10p_R^2 + \frac{33}{50}p_X^2 \right). \quad [S16]$$

111 The candidate extremals for the optimal control problem are now obtained as solutions to the following Hamiltonian dynamics

$$112 \quad \dot{\Psi} = \frac{\partial H}{\partial p}, \quad \dot{p} = -\frac{\partial H}{\partial \Psi}. \quad [S17]$$

113 For the state variables $\Psi(t)$, this gives back Eq. S14 driven by the optimal controls $(\zeta_0^*(t), \Delta\zeta^*(t))$. Translational invariance
 114 enforces that $\partial_X \mathcal{H} = 0$, hence p_X is also conserved along the optimal trajectory.

115 To obtain analytical expressions for the costate and optimal activity, we reparametrize time to linearize the dynamics. We
 116 define $\tau(t)$ so that

$$117 \quad \dot{\tau} = \frac{1}{\eta R^2}, \quad [S18]$$

118 with the initial condition $\tau(0) = 0$. Upon writing $\chi(\tau) = X(t)$, $\rho(\tau) = R(t)$, $P(\tau) = p_R(t)$ and $p_X = p_0$ (a constant), we obtain
 119 (primes denote d/d τ)

$$120 \quad \chi' = \frac{33}{175}p_0, \quad \rho' = \frac{20}{7}P, \quad P' = 2\eta\rho H_0, \quad [S19]$$

121 where H_0 (constant) is the conserved value of the Hamiltonian along the optimal solution. The position equation can be
 122 integrated trivially to give $\chi(\tau) = (33p_0/175)\tau$ and the last two equations can be combined to give a linear equation for $\rho(\tau)$,

$$123 \quad \rho'' - \kappa^2 \rho = 0 \implies \rho(\tau) = \frac{R_T \sinh(\kappa\tau) + R_0 \sinh[\kappa(\tau(T) - \tau)]}{\sinh[\kappa\tau(T)]}, \quad [S20]$$

124 where we have used the boundary conditions on the drop size ($R(0) = R_0$, $R(T) = R_T$) and defined the (as of yet) undetermined
 125 constant

$$126 \quad \kappa^2 = \frac{40\eta}{7} H_0. \quad [S21]$$

127 By imposing the boundary condition on the drop position, we obtain

$$128 \quad X_T = \frac{33}{175}p_0\tau(T). \quad [S22]$$

129 To compute the value of κ and p_0 , we evaluate the conserved Hamiltonian at $t = 0$ to get

$$130 \quad H_0 = \frac{1}{7\eta R_0^2} \left[10p_R(0)^2 + \frac{33}{50}p_0^2 \right], \quad [S23]$$

131 in which we can plug in the value of $p_R(0) \equiv P(0) = (7/20)\rho'(0) = (7\kappa/20 \sinh[\kappa\tau(T)])\{R_T - R_0 \cosh[\kappa\tau(T)]\}$ along with the
 132 expression for p_0 (from Eq. S22) to get

$$133 \quad \frac{X_T}{R_0} = \sqrt{\frac{33s^2}{500 \sinh^2 s} \left[\left(\frac{R_T}{R_0} \right) \left(2 \cosh s - \frac{R_T}{R_0} \right) - 1 \right]} \equiv \mathcal{F} \left(s, \frac{R_T}{R_0} \right), \quad [S24]$$

134 where $s = \kappa\tau(T) > 0$ is the unknown to be solved for. Remarkably, this equation lacks any real solution for $s > 0$ if X_T/R_0 is
 135 sufficiently large for a given ratio R_T/R_0 . One can show that Eq. S24 in general has either two positive solutions, one positive
 136 solution or none. By using the locus of coinciding solutions, we can compute the envelope curve of the maximal displacement
 137 for a given size change, above which Eq. S24 has no real solutions. This determines the feasibility curve for smooth optimal

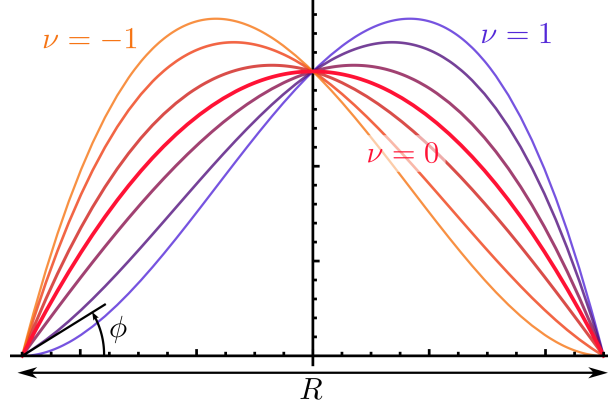


Fig. S1. The parametrization of the drop height (Eq. S27) in terms of its position (X , here centered to the origin), size (R) and an asymmetry (ν) that captures a tilt in the drop profile. For $\nu = 0$, we have a symmetric parabolic drop, and for $\nu = \pm 1$ (the extremes), we have a right or left leaning drop respectively. The asymmetry ν is related to the trailing (left most) contact angle ϕ of the drop, via $\nu = 1 - (R^2\phi/6)$.

138 policies (at least C^2 regularity) which is shown in Fig. 3C (main text). For large size disparities $R_T/R_0 \rightarrow \infty$, we can compute
 139 the asymptotic behaviour of this bounding curve to be a weak logarithm,

$$140 \quad \left(\frac{X_T}{R_0}\right)_{\max} \simeq \sqrt{\frac{33}{500}} \ln\left(\frac{2R_T}{R_0}\right) \quad \left(\frac{R_T}{R_0} \rightarrow \infty\right). \quad [\text{S25}]$$

141 By solving Eq. S24 numerically for $s = \kappa\tau(T)$, when it exists, we can use the final relation between $\tau(T)$ and T (by integrating
 142 Eq. S18) to get

$$143 \quad \frac{\tau(T)}{4s \sinh^2 s} [(R_0^2 + R_T^2)(\sinh(2s) - 2s) + 4R_0R_T(s \cosh s - \sinh s)] = \frac{T}{\eta}. \quad [\text{S26}]$$

144 This directly gives $\tau(T)$ and hence κ (by using the now known value of s) and also H_0 , thereby completing the full solution.
 145 Representative curves for the control and state dynamics are plotted in Fig. 2A-B, main text.

146 **C. Three parameter model reduction.** Here we extend our previous calculation to include a third variable that captures changes
 147 in drop shape, in addition to position and size. We parametrize the shape of the drop using a cubic polynomial that additionally
 148 captures the drop asymmetry, i.e., a tilting of the drop (see Fig. S1). Upon using $\int dx h = 1$, we obtain

$$149 \quad h(x, t) = \frac{12\nu(t)}{R(t)^4} (x - a(t))(a(t) + R(t) - x) \left[x - a(t) + \frac{R(t)(1 - \nu(t))}{2\nu(t)} \right], \quad [\text{S27}]$$

where $a(t) = X(t) - (R(t)/2) - R(t)\nu(t)/10$ is the trailing (left most) end of the drop and the dimensionless asymmetry
 $\nu \in [-1, 1]$ is related to the trailing (left most) contact angle $\phi(t)$ of a right moving drop via $\nu(t) = 1 - R(t)^2\phi(t)/6$ (see
 Fig. S1). Note that $h = 0$ at the two ends of the drop, $x = a$ and $x = a + R$, and vanishes outside this region. The drop
 asymmetry is restricted to the interval $\nu \in [-1, 1]$ to ensure the height in Eq. S27 is always positive. While the extreme limits
 of $\nu = \pm 1$ correspond to a right and left leaning drop respectively (Fig. S1), for $\nu = 0$, the drop has a symmetric parabolic
 profile. The position $X = \int dx xh$ is still directly given by the mean, while the size $R(t)$ and asymmetry $\nu(t)$ of the drop can
 be related to the next two spatial moments of the height field $h(x, t)$ via

$$\Delta = \int dx (x - X)^2 h = \frac{R^2}{100} [5 - \nu^2], \quad [\text{S28}]$$

$$\Phi = \int dx (x - X)^3 h = \frac{R^3}{3500} \nu [7\nu^2 - 15]. \quad [\text{S29}]$$

We can similarly compute spatial moments of the flux $q(x, t)$ to obtain

$$\dot{X}(t) = \int_{a(t)}^{a(t)+R(t)} dx q(x, t), \quad [\text{S30}]$$

$$\dot{\Delta}(t) = 2 \int_{a(t)}^{a(t)+R(t)} dx (x - X(t)) q(x, t), \quad [\text{S31}]$$

$$\dot{\Phi}(t) = 3 \int_{a(t)}^{a(t)+R(t)} dx [(x - X(t))^2 - \Delta(t)] q(x, t). \quad [\text{S32}]$$

150 These equations can be inverted using Eqs. S28, S29 to obtain the complete and closed set of nonlinear dynamical equations
 151 governing the evolution of the drop position $X(t)$, size $R(t)$ and tilt $\nu(t)$. If we set $\nu = 0$ by fiat and neglect $\dot{\nu}$, then we recover

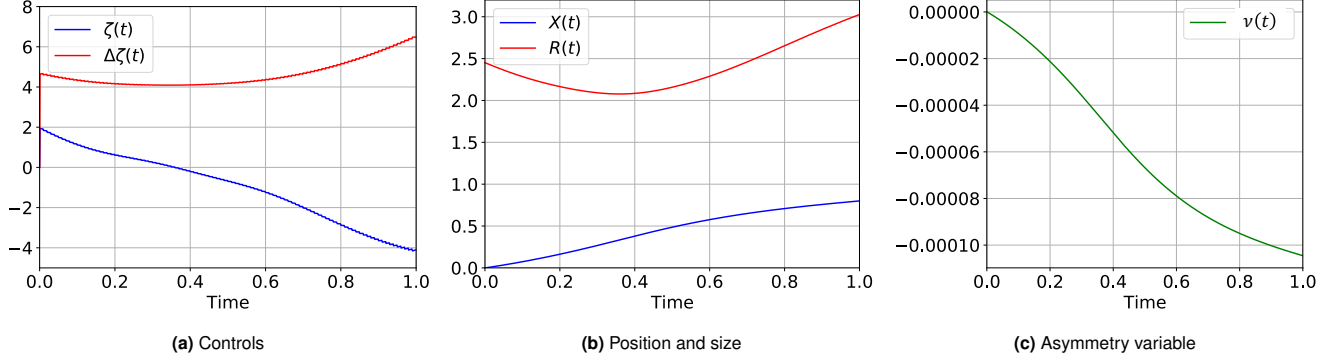


Fig. S2. Solution to the three dimensional ODE optimal control problem with the asymmetry variable ($\nu(t)$) forced to be close to zero (when $\lambda = 10^5$) recovers the gather-move-spread strategy. The optimal control problem was solved using CasADi (7) with time step $dt = 10^{-3}$. The final position was imposed as a hard terminal constraint, while the size constraint was imposed by a quadratic terminal cost with $\mu_R = 9 \times 10^{-6}$.

152 the coupled (X, R) dynamics given in Eq. S14. Note that, the low order Galerkin approximation employed here works best in
 153 the bulk of the drop and cannot capture boundary phenomena that occur close to the contact line. While the symmetric drop
 154 parametrized by just X, R (Eq. S12) corresponds to an effective zero mode of capillary forces (atleast within the bulk of the
 155 drop, size change is affected by γ when the contact angle and boundary effects are taken into account), in the presence of drop
 156 asymmetry ($\nu \neq 0$), this is no longer the case and surface tension forces will be present even in the bulk of the drop. In order
 157 to avoid complications from surface tension and wetting related boundary effects, we set $\gamma = 0$ for simplicity here and consider
 158 only the high active capillary number limit ($Ca_\zeta \gg 1$).

159 **D. Three dimensional ODE optimal control.** The configuration space of the drop is now characterized by a three dimensional
 160 manifold \mathcal{M} comprising the state variables X, R and ν . We denote the state by a column vector $\Psi(t) = [X(t) R(t) \nu(t)]^T$
 161 evolving in $\mathcal{M} = \mathbb{R} \times \mathbb{R}^+ \times [-1, 1]$, whose dynamics constitutes a time-invariant, drift-free, and underactuated nonlinear
 162 dynamical system, given by

$$\dot{\Psi}(t) = \zeta_0(t)F(\Psi(t)) + \Delta\zeta(t)G(\Psi(t)), \quad [\text{S33}]$$

where $F(\Psi), G(\Psi)$ are nonlinear control vector fields, constituting two tangent directions that are independently controlled by
 the mean and gradient components of the activity

$$F(\Psi) = \frac{1}{\eta R^4} \begin{bmatrix} 0 \\ F_R \\ F_\nu \\ R \end{bmatrix} = \frac{1}{\eta R^4} \begin{bmatrix} 0 \\ \frac{24[224 - (1 - \nu^2)(448 - 81(1 - \nu^2))]}{1001(1 - \nu^2)} \\ \frac{1}{\lambda R} \frac{768\nu[28 + (1 - \nu^2)(14 - 3(1 - \nu^2))]}{1001(1 - \nu^2)} \\ R \end{bmatrix}, \quad [\text{S34}]$$

$$G(\Psi) = \frac{1}{\eta R^4} \begin{bmatrix} G_X \\ G_R \\ G_\nu \\ R \end{bmatrix} = \frac{1}{\eta R^4} \begin{bmatrix} \frac{18}{5005} [224 - 3(1 - \nu^2)(28 - (1 - \nu^2))] \\ -\frac{12\nu[352 - (1 - \nu^2)(80 - (1 - \nu^2))]}{1001(1 - \nu^2)} \\ \frac{1}{\lambda R} \frac{24[704 + (1 - \nu^2)(32 - (1 - \nu^2)(104 - 5(1 - \nu^2)))]}{1001(1 - \nu^2)} \\ R \end{bmatrix}. \quad [\text{S35}]$$

164 Note that elements of $F(\Psi)$ and $G(\Psi)$ depend only on the size and asymmetry of the drop and are independent of its position,
 165 as expected from translational invariance. Interestingly though, F doesn't contribute to \dot{X} , i.e., even when the drop is spatially
 166 asymmetric ($\nu \neq 0$), a mean activity ($\zeta_0 \neq 0$) does not generate translation, although this is not prohibited by the Curie
 167 principle (8). We have introduced an additional continuation parameter λ in the dynamics of ν alone, that we tune from
 168 $\lambda = \infty$ to $\lambda = 1$ within a homotopy continuation scheme. In the $\lambda = \infty$ limit, we recover the symmetric drop system for which
 169 we have exact analytical optimal controls provided in Sec. B. Upon decreasing λ with concurrent numerical optimization using
 170 CasADi (7), we progressively deform the known symmetric drop optimal solution into the required optimal policy for the three
 171 parameter problem when $\lambda = 1$. As before, we assume the initial location of the drop is $X_0 = 0$, and initial size $R_0 = \sqrt{6}$. The
 172 final position and size are also fixed to be $X_T = 0.8$ and $R_T = 3$. While we impose a fixed end-point condition on the position
 173 ($X(T) = X_T$), we relax the terminal constraint on size with a finite cost $\mu_R = 9 \times 10^{-6}$ and leave $\nu(T)$ as a free, unconstrained
 174 variable.

175 We verify controllability symbolically using Mathematica (9) by checking that $\{F, G, [F, G]\}$ constitute a basis spanning the
 176 tangent space everywhere in the drop configuration space, $\text{int}(\mathcal{M}) = \mathbb{R} \times (0, \infty) \times (-1, 1)$. Hence Eq. S33 is controllable and
 177 presents an underactuated sub-Riemannian optimal control problem. This implies that there exist controls $(\zeta_0, \Delta\zeta)$ that can
 178 steer any state Ψ_1 into any other state Ψ_2 in the drop configuration space.

179 The net dissipation is once again $\mathcal{W} = \int_0^T dt \mathcal{L}$, with the Lagrangian

$$180 \quad \mathcal{L} = \frac{1}{\eta R^6} \mathbf{u}^T \mathbf{M} \mathbf{u}, \quad [\text{S36}]$$

where $\mathbf{u} = [\zeta \ \Delta\zeta]^T$ and \mathbf{M} is a 2×2 symmetric matrix with entries

$$M_{11}(\nu) = \frac{72}{385} [32 - 3(1 - \nu^2)(8 - (1 - \nu^2))] , \quad [\text{S37}]$$

$$M_{12}(\nu) = -\frac{72}{10010} \nu [160 - 3(1 - \nu^2)(24 - (1 - \nu^2))] , \quad [\text{S38}]$$

$$M_{22}(\nu) = \frac{18}{5005} [416 - (1 - \nu^2)(248 - 27(1 - \nu^2))] . \quad [\text{S39}]$$

One can easily check by explicit diagonalization that \mathbf{M} is always positive definite and nondegenerate ($\det(\mathbf{M}) \neq 0$) for all $\nu \in [-1, 1]$. Compactly, the dynamics can be written as $\dot{\Psi} = \mathbf{\Omega}(\Psi)\mathbf{u}$, where

$$\begin{aligned} \mathbf{\Omega}(\Psi) &= [F \ G] \\ &= \frac{1}{\eta R^6} \begin{bmatrix} 0 & R^2 G_X \\ R^2 F_R & R^2 G_R \\ R F_\nu & R G_\nu \end{bmatrix} . \end{aligned} \quad [\text{S40}]$$

Further, let $V_1 = R p_R F_R + p_\nu F_\nu$ and $V_2 = R p_X G_X + R p_R G_R + p_\nu G_\nu$, and $\tilde{\mathbf{v}} = [V_1 \ V_2]^T$ so that $\mathbf{w} = R\tilde{\mathbf{v}}$. The pre-Hamiltonian $\mathcal{H}(\Psi, p, \mathbf{u})$, and the optimal control \mathbf{u}^* obtained by its maximization is given as

$$\mathcal{H}(\Psi, p, \mathbf{u}) = \frac{1}{\eta R^6} [\mathbf{w}^T \mathbf{u} - \mathbf{u}^T \mathbf{M} \mathbf{u}] , \quad [\text{S41}]$$

$$\mathbf{u}^* = \frac{1}{2} \mathbf{M}^{-1} \mathbf{w} , \quad [\text{S42}]$$

181 so that the Hamiltonian is

$$182 \quad H(\Psi, p, u^*) = \frac{1}{4\eta R^6} \mathbf{w}^T \mathbf{M}^{-1} \mathbf{w} . \quad [\text{S43}]$$

From this, we can write the co-state equations as

$$\dot{p}_X = 0 , \quad [\text{S44}]$$

$$\begin{aligned} \dot{p}_R &= -\frac{\partial H}{\partial R} \\ &= -\left\{ -\frac{6H}{R} + \frac{1}{4\eta R^6} 2\mathbf{w}^T \mathbf{M}^{-1} \frac{\partial \mathbf{w}}{\partial R} \right\} \\ &= \frac{6H}{R} - \frac{1}{2\eta R^6} \mathbf{w}^T \mathbf{M}^{-1} \left[\tilde{\mathbf{v}} + R \frac{\partial \tilde{\mathbf{v}}}{\partial R} \right] , \end{aligned} \quad [\text{S45}]$$

where $\frac{\partial \tilde{\mathbf{v}}}{\partial R} = [p_2 F_R \ p_1 G_X + p_2 G_R]^T$. Moreover,

$$\begin{aligned} \dot{p}_\nu &= -\frac{\partial H}{\partial \nu} \\ &= -\frac{1}{4\eta R^6} \left\{ 2\mathbf{w}^T \mathbf{M}^{-1} \frac{\partial \mathbf{w}}{\partial \nu} + \mathbf{w}^T \frac{\partial \mathbf{M}^{-1}}{\partial \nu} \mathbf{w} \right\} . \end{aligned} \quad [\text{S46}]$$

183 We find the solution to the optimal transport problem by using CasADi (7). A fourth order Runge-Kutta method is used to
 184 simulate the dynamics, using time step $dt = 10^{-3}$. As the optimal control solutions scale with the viscosity, we set $\eta = 0.1$ for
 185 computational convenience, without loss of generality. Since the initialization of the solver affects the optimal control solution
 186 obtained, we obtain the optimal solution presented in Fig. S3 from initial guesses for the controls: two optimal solutions for the
 187 symmetric drop and 100 randomized initial values of the control parameters at each time of discretization, uniformly sampled
 188 in the interval $[-0.5, 0.5]$. The mean stress becomes activated during earlier times causing non-monotonic changes in the drop
 189 size while the gradient component remains close to zero, where during later durations of the transport, the gradient stress
 190 drives the drop to its prescribed final location, while the shape undergoes significant change from that of a symmetric drop to a
 191 highly asymmetric one. As expected, in the limit of $\lambda \rightarrow \infty$ ($\dot{\nu}, \nu \rightarrow 0$), the protocol for the optimal transportation of the drop
 192 recovers the analytical strategy described for the symmetric drop (Fig. S2).

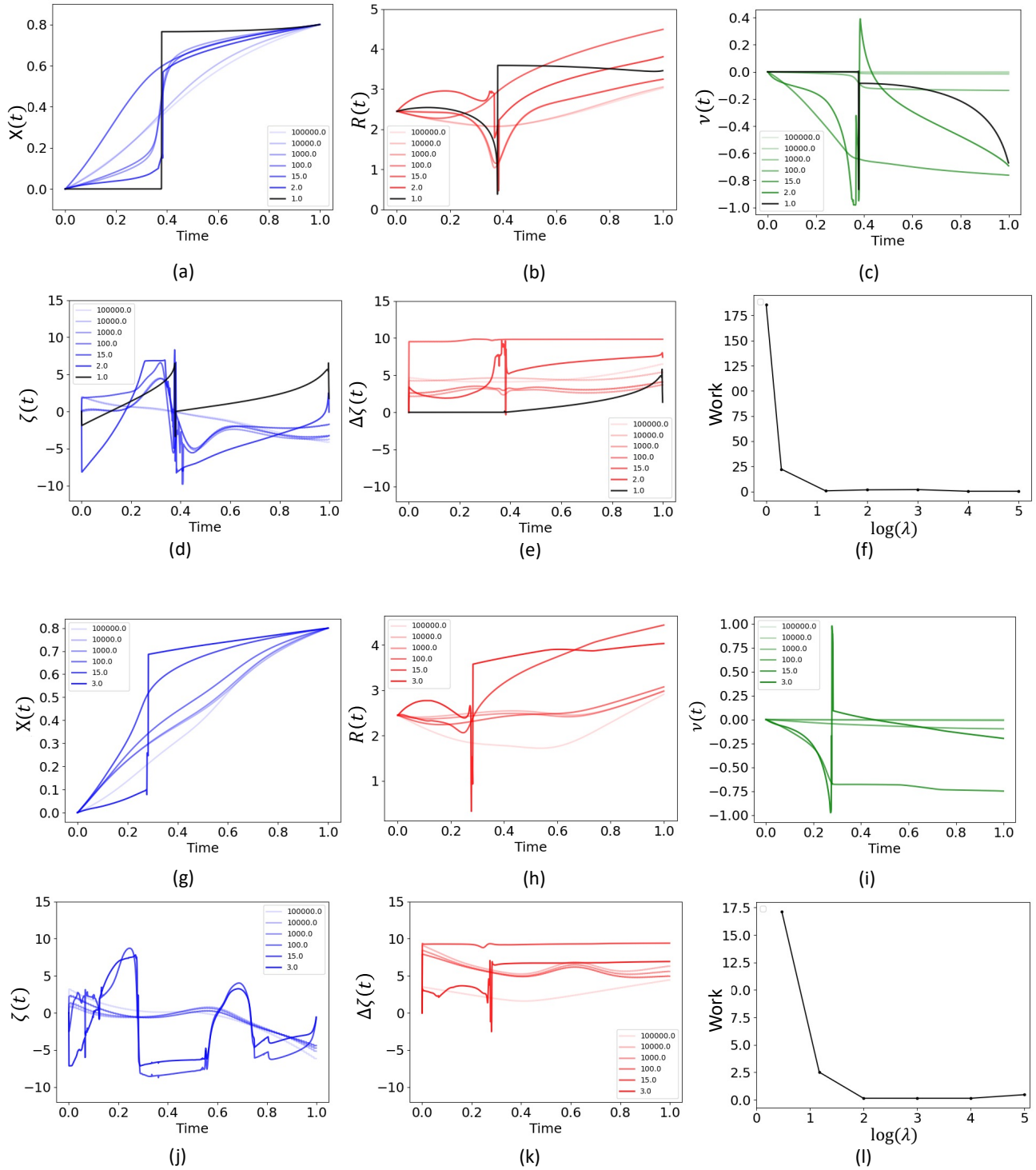


Fig. S3. Sensitivity of the optimal control solution to the parameter λ in the equation for asymmetry variable ν . The optimal solutions to the drop transport using three dimensional ODE model are obtained via sequentially optimizing while varying λ from 10^5 to 1, with the solver at the first iteration initialized by the solution to the optimal transport for the symmetric drop. The optimal control problems were solved in CasADi with time step 0.001. Plots (a)-(f) and (g)-(l) respectively show the solution to the sequential optimization problem starting with the two symmetric solutions from the two dimensional ODE optimal control problem. While the algorithm converges for initialization based on the first symmetric solution when $\lambda = 1$ corresponding to the asymmetric drop model, it does not for the second solution. We observe that in both cases for smaller values of λ , the drop undergoes rapid changes in size (R) and shape (ν), thereby affecting the numerical stability of the optimization.

193 3. PDE optimal control

194 The numerical optimization with the PDE constraint is carried out with the evolutionary algorithm CMA-ES (10), using a
 195 standard Python implementation called pycma (11). The strategy involves solving a large number of forward problems with
 196 randomly sampled controls and evolving the population to optimize a given cost, which dictates the fitness landscape. We
 197 discretize and solve the PDE as a forward problem using the finite element method in FEniCS (12, 13). We choose the domain
 198 to be an interval of length $L = 8$ with a step size $dx \approx 0.01$ (resolution $N = 800$). We use a mixed semi-implicit scheme treating
 199 the nonlinear mobility explicitly using a second order Adams-Bashforth method and the linear stress tensor via Crank-Nicolson.
 200 Upon choosing $dt = 5 \times 10^{-3}$ as the integration time-step and a total time $T = 1$, we use the following weak form of the PDE,

$$201 \left\langle \left(\frac{h_{n+1} - h_n}{dt} \right), v_1 \right\rangle - \langle q_{n+1/2}, \partial_x v_1 \rangle = 0, \quad [S47]$$

202 where $q_{n+1/2}$ is the flux evaluated in the middle of the time step and the inner product is $\langle f, g \rangle = \int dx fg$. We use second order
 203 Lagrange elements for our function space basis. To avoid solving a nonlinear equation, we evaluate the flux in a semi-linear
 204 fashion as

$$205 q_{n+1/2} = \Gamma(h_*) [\partial_x \sigma(h_{n+1/2}) - \Pi'(h_*) \partial_x h_{n+1/2}], \quad [S48]$$

206 where $h_* = (3h_n - h_{n-1})/2$, $h_{n+1/2} = (h_n + h_{n+1})/2$. The mobility Γ is regularized to preserve positivity of the solution (given
 207 a compliant initial condition) as (14, 15)

$$208 \Gamma(h) = \frac{h^4 m(h)}{h^4 + \epsilon m(h)}, \quad m(h) = \frac{h^3}{3\eta}, \quad [S49]$$

with $\epsilon = 10^{-8}$ and viscosity $\eta = 0.1$. The stress σ includes both active and passive (capillary) contributions, with the latter
 providing numerical stability as well,

$$\sigma(h) = \zeta h + \gamma \mu, \quad [S50]$$

$$\langle \mu, v_2 \rangle + \langle \partial_x h, \partial_x v_2 \rangle = 0. \quad [S51]$$

209 Both test functions, v_1 and v_2 belong to the same function space spanned by second order Lagrange elements. Hence we solve
 210 for the higher gradient term $\mu = \partial_x^2 h$ in weak form simultaneously with the continuity equation above. The constant surface
 211 tension is varied over the range $\gamma = 0.075 - 2$. We also include a disjoining pressure $\Pi(h)$ (16) to fix the contact angle for a
 212 passive, sessile drop. Along with a precursor film of thickness $\delta = 10^{-2}$, we have (17–19)

$$213 \Pi(h) = \frac{\mathcal{A}}{h^3} \left(1 - \frac{\delta}{h} \right), \quad \Pi'(h) = \frac{\mathcal{A}}{h^5} (4\delta - 3h), \quad [S52]$$

214 where $\mathcal{A} = 3\gamma\delta^2 \tan^2 \phi_0$, where ϕ_0 is the required contact angle. This expression works better than $6\gamma\delta^2(1 - \cos \phi_0)$ (19) to
 215 which it is equivalent for small angles ($\phi_0 \ll 1$). We set $\phi_0 = \pi/4$ to be the equilibrium contact angle.

216 The boundary conditions are $\partial_x h = 0$ and $M\partial_x \sigma = 0$ at the ends of the domain (not the drop). Note, we do not track
 217 the contact line separately. Mass is naturally conserved within this formulation, which we check numerically as well. We fix
 218 $\int dx h(x) = 1$ with the integral spanning the whole domain. The initial condition is a symmetric drop centered around the
 219 origin $X_0 = 0$ with size R_0 ,

$$220 h(x) = \begin{cases} \delta + \frac{6}{R_0^3} [1 - R_0\delta] \left(\frac{R_0^2}{4} - x^2 \right), & x \in \left[-\frac{R_0}{2}, \frac{R_0}{2} \right] \\ \delta, & x \notin \left[-\frac{R_0}{2}, \frac{R_0}{2} \right] \end{cases}. \quad [S53]$$

221 The initial size is fixed by the equilibrium contact angle to be $R_0 = \sqrt{6/\tan \phi_0} = \sqrt{6}$. We checked that this initial condition is
 222 the stable steady state of the passive dynamics if the activity is turned off.

223 The activity ζ is taken to be a linear profile in space

$$224 \zeta(x, t) = \zeta_1(t) + \zeta_2(t)x. \quad [S54]$$

Note that this is in the lab frame over the entire domain and not in the drop fixed comoving frame (Eq. 4, main text). While
 the latter is more convenient to use for the analytical calculations, it is numerically easier to use the entirely equivalent lab
 fixed parametrization. The mean and gradient activity ($\zeta_0, \Delta\zeta$) and the drop position and size (X, R) are computed using

$$\zeta_0 = \frac{1}{R} \int dx \zeta \Theta(h - \delta_c), \quad \Delta\zeta = \zeta_2 R \quad [S55]$$

$$X = \int dx x h, \quad R = \int dx \Theta(h - \delta_c), \quad [S56]$$

225 where $\Theta(x)$ is the Heaviside step function, and $\delta_c = 1.1\delta = 1.1 \times 10^{-2}$ is a threshold cutoff to separate the prewetting film
 226 from the drop. We use a real element with one global degree of freedom to represent ζ_1 and ζ_2 and choose to discretize the

control with a coarser time-step than required for a numerically stable integration of the PDE. This keeps the optimization manageable and avoids excessively high-dimensional searches. The control is actuated at $N_c = 100$ equi-spaced time points, and is linearly interpolated in between.

The cost function $\mathcal{C} = \mathcal{W} + \mathcal{T} + \mathcal{R}$ involves a time integrated cost from the net dissipated energy (\mathcal{W}), a terminal cost (\mathcal{T}) and a regularizing term (\mathcal{R}), which are given by

$$\mathcal{W} = \int_0^T dt \int dx \frac{h^3}{3\eta} [\partial_x(\sigma - \Pi)]^2, \quad [\text{S57}]$$

$$\mathcal{R} = \frac{\alpha}{N_c} \int_0^T dt [(\partial_t \zeta_1)^2 + (\partial_t \zeta_2)^2], \quad [\text{S58}]$$

$$\mathcal{T} = \mu_X \left(\frac{X(T) - X_T}{X_T} \right)^2 + \mu_R \left(\frac{R(T) - R_T}{R_T} \right)^2. \quad [\text{S59}]$$

We use a trapezoidal rule to discretize the temporal integral in \mathcal{W} and the spatial integral is computed using standard Gauss quadrature in FEniCS. The temporal regularization term \mathcal{R} is computed easily by noting that the control is piecewise linear. Note that, $\mathcal{R} \rightarrow 0$ as $N_c \rightarrow \infty$ for fixed α and bounded time derivatives of the activity. The regularization term is used to select smoother controls and is reminiscent of minimal attention control (20). If the total cost is infinite or negative, or if $h < 0$ at any spatial point at any time, we set it to NaN, forcing CMA-ES to discard the run and reevaluate it for a random sampling of activity. We fix the following parameters,

$$X_0 = 0, \quad R_0 = \sqrt{6}, \quad R_T = 3.0, \quad \eta = 0.1, \quad T = 1, \quad \mu_X = \mu_R = 10^3, \quad [\text{S60}]$$

and vary both the surface tension ($\gamma = 0.075 - 2$) and the terminal drop position ($X_T = 0.6, 0.8, 1.0, 1.2$). For the parameters chosen, both $X_T = 0.6, 0.8$ correspond to transport tasks for which we have a symmetric solution, while for $X_T = 1.0, 1.2$, the symmetric solution doesn't exist. We use multiple initializations for the optimization routine, including the two solutions (one global and one local optimum) we obtain from the symmetric problem (for the parameters where this is unavailable, we use the closest available symmetric solutions using $X_T = 0.9, R_T = 3$ instead), two independent sets of random activity and a sequential minimization using the best solution obtained. The random activity initializations are uniformly sampled from the interval $[-\zeta_{\max}/4, \zeta_{\max}/4]$, where $\zeta_{\max} = 20$ is the maximum permitted value of the activity to avoid numerical blow-up. We use an initial standard deviation $\Sigma_{\text{dev}} = 0.2$ for both ζ_1 and ζ_2 , along with a default population size of $N_{\text{pop}} = 19$. The repeated function calls over the random population are parallelized over N_{pop} cores using Python's multiprocessing pool module. The maximum number of iterations is fixed at $M_{\text{iter}} = 10^6$ and we set the convergence criteria to be

$$\Delta \mathcal{C} \leq \epsilon_{\text{tol}} (C_{\text{median}}^0 - C_{\text{median}}^{\min}), \quad [\text{S61}]$$

where $\Delta \mathcal{C} = \max(\mathcal{C}) - \min(\mathcal{C})$ is the current spread in the fitness function, C_{median}^0 is the median of the initial fitness distribution and C_{median}^{\min} is the smallest median fitness encountered throughout the optimization trajectory. We choose $\epsilon_{\text{tol}} = 10^{-4}$.

4. Minimal dissipation bound

Here we derive the general bound on the minimal amount of energy that must be dissipated if a drop with bounded height achieves a nonzero displacement and size change. Consider a positive and bounded height function $h(x, t) \geq 0$ with compact nonvanishing support ($\forall t \in [0, T]$) and net unit mass ($\int dx h = 1$) obeying the continuity equation

$$\partial_t h + \partial_x q = 0, \quad q = h \langle u \rangle = \frac{h^3}{3\eta} \partial_x \sigma, \quad [\text{S62}]$$

driven by some arbitrary stress $\sigma = \sigma(x, t, h, \partial_x h, \dots)$. Note, we do not specify the rheological constitutive equation for σ , nor the form of the control, so the description is entirely general to bulk stress driven drop motion*. The net dissipation in the drop is simply

$$\mathcal{W} = \int_0^T dt \int dx h \langle u \rangle \partial_x \sigma = 3\eta \int_0^T dt \int dx \frac{q^2}{h^3}, \quad [\text{S63}]$$

once again irrespective of the constitutive relation for σ . From Eqs. S30 and S31, we know that $\dot{X} = \int dx q$ and $\dot{\Delta} = 2 \int dx (x - X) q$, where the position is $X = \int dx x h$ and the variance is $\Delta = \int dx (x - X)^2 h$. These are all finite as h is assumed to have compact support at all times. Upon integrating \dot{X} and using $X(0) = X_0 = 0$ without loss of generality, we obtain,

$$|X(T)| = \left| \int_{t,x} \left(\frac{q}{h} \right) h \right| \leq \sqrt{\int_{t,x} \frac{q^2}{h^2} \int_{t,x} h^2}, \quad [\text{S64}]$$

where $\int_{t,x} = \int_0^T dt \int dx$ and we have used the Cauchy-Schwarz inequality. We can now use Hölder's inequality to write

$$\int_{t,x} h^2 \leq \|h\|_{\infty} \int_{t,x} h = T \|h\|_{\infty}, \quad [\text{S65}]$$

*Drop motion driven by marangoni forces or differential surface wetting are not included in this formalism as they appear as body forces and not stresses in lubrication theory.

265 where $\|h\|_\infty = \sup_{x,t} h(x,t)$ is the maximum height achieved by the drop at any point along its trajectory (guaranteed to be
 266 finite by mass conservation and the nonvanishing compact support). Similarly, we can use Eq. S63 and Hölder’s inequality
 267 again to write

$$268 \int_{t,x} \frac{q^2}{h^2} \leq \|h\|_\infty \int_{t,x} \frac{q^2}{h^3} = \|h\|_\infty \frac{\mathcal{W}}{3\eta}. \quad [\text{S66}]$$

269 Upon combining Eqs. S64-S66 we get

$$270 X(T)^2 \leq \|h\|_\infty^2 \frac{T\mathcal{W}}{3\eta}. \quad [\text{S67}]$$

271 We can perform a similar calculation for the variance Δ . This gives (using the Cauchy-Schwarz inequality)

$$272 |\Delta(T) - \Delta(0)| = 2 \left| \int_{t,x} q(x-X) \right| \leq 2 \sqrt{\int_{t,x} \frac{q^2}{h^2} \int_{t,x} h^2(x-X)^2}. \quad [\text{S68}]$$

273 Hölder’s inequality can now be used to simplify the right hand side,

$$274 \int_{t,x} h^2(x-X)^2 \leq \|h\|_\infty \int_0^T dt \Delta(t) = \|h\|_\infty T \langle \Delta \rangle_T, \quad [\text{S69}]$$

275 with $\langle \Delta \rangle_T$ as the time averaged variance of the drop (also finite). This gives

$$276 |\Delta(T) - \Delta(0)|^2 \leq 4 \frac{T\mathcal{W}}{3\eta} \|h\|_\infty^2 \langle \Delta \rangle_T. \quad [\text{S70}]$$

277 By combining Eqs. S67, S70, we obtain the desired lower bound on the dissipation to be

$$278 \mathcal{W}_{\min} \equiv \frac{3\eta}{2T\|h\|_\infty^2} \left[X(T)^2 + \frac{|\Delta(T) - \Delta(0)|^2}{4\langle \Delta \rangle_T} \right] \leq \mathcal{W}. \quad [\text{S71}]$$

279 5. Supplementary Movies

280 **Movie S1.** Video showing the activity controls $\{\zeta_0(t), \Delta\zeta(t)\}$, the state trajectory $\{X(t), R(t)\}$ and full drop
 281 profile $h(x,t)$ corresponding to the optimal solution computed using CMA-ES for a large active capillary
 282 number ($\text{Ca}_\zeta = 383.66$), or conversely small surface tension ($\gamma = 0.15$). The optimal policy qualitatively
 283 employs a “gather-move-spread” strategy, though now with complex shape changes of the drop. The drop
 284 is initialized as a parabola centered at $X_0 = 0$, with size $R_0 = \sqrt{6}$ (equilibrium contact angle $\phi_{\text{eq}} = \pi/4$) with
 285 terminal position and size fixed to $X_T = 0.8$ and $R_T = 3$ respectively. The viscosity ($\eta = 0.1$) and the total time
 286 ($T = 1$) are fixed as well.

287 **Movie S2.** Video showing the activity controls $\{\zeta_0(t), \Delta\zeta(t)\}$, the state trajectory $\{X(t), R(t)\}$ and full drop
 288 profile $h(x,t)$ corresponding to the optimal solution computed from the reduced order ODE model in Eq. 10.
 289 The activity profile plotted in Fig. 3A is used as the input for the simulation and the surface tension is set
 290 to a small value ($\gamma = 0.15$) as appropriate for the validity of the ODE model. All other parameters are kept
 291 the same ($X_0 = 0$, $R_0 = \sqrt{6}$, $X_T = 0.8$, $R_T = 3$, $\eta = 0.1$, $T = 1$) The drop trajectory is qualitatively similar to the
 292 full optimal solution (shown in Movie S1 and Fig. 4A) but not quantitatively accurate, for instance, the final
 293 drop position and size only reach about half the values set by the task.

294 **Movie S3.** Video showing the activity controls $\{\zeta_0(t), \Delta\zeta(t)\}$, the state trajectory $\{X(t), R(t)\}$ and full drop
 295 profile $h(x,t)$ corresponding to the optimal solution computed using CMA-ES for a small active capillary
 296 number ($\text{Ca}_\zeta = 30.91$), or conversely small surface tension ($\gamma = 2$). The optimal policy is now quite different,
 297 with futile size oscillations that initially dissipate energy without translation and a final activity burst that
 298 advances the drop at the end of the policy. The drop is initialized as a parabola centered at $X_0 = 0$, with size
 299 $R_0 = \sqrt{6}$ (equilibrium contact angle $\phi_{\text{eq}} = \pi/4$) and the terminal position and size are $X_T = 0.8$ and $R_T = 3$
 300 respectively. The viscosity ($\eta = 0.1$) and the total time ($T = 1$) are fixed as well.

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